

MODELING AND NUMERICAL STUDY OF NONSMOOTH DYNAMICAL SYSTEMS.

Applications to Mechanical and Power Electronic Systems.

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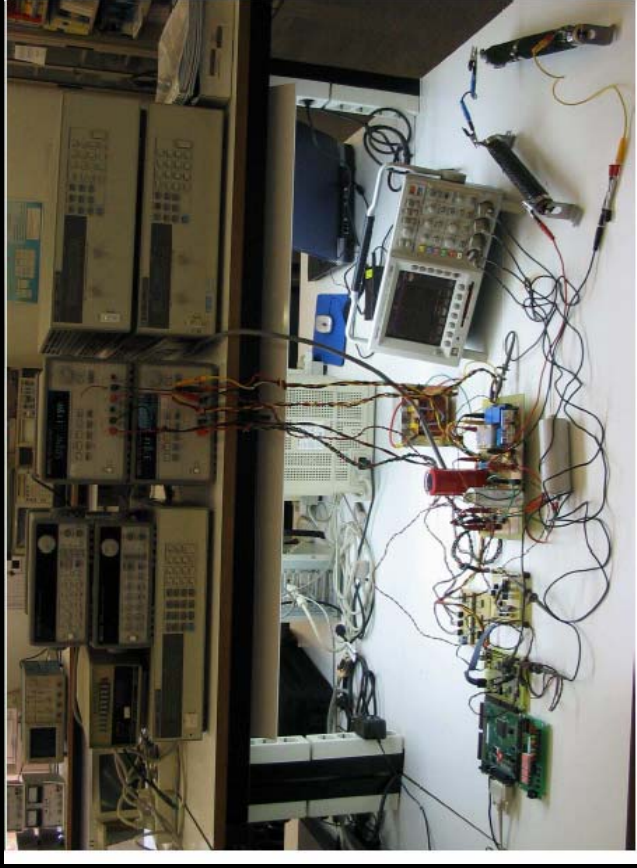
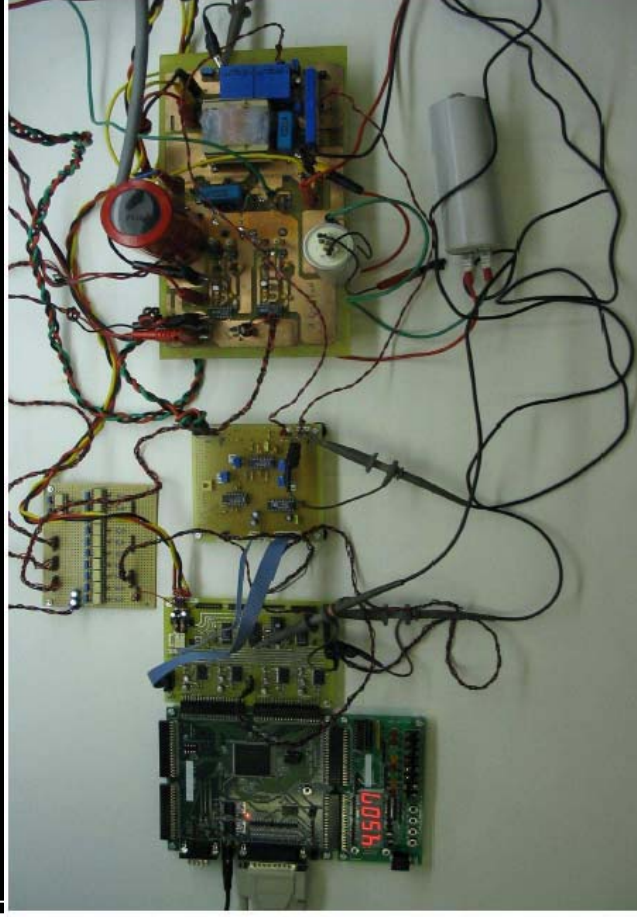
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Thesis

Sevilla-Islantilla, October 18-21, 2006

Introduction and motivation

■ Why nonsmooth?



NOT EVERYTHING IS COMPLETELY SMOOTH

Introduction and motivation

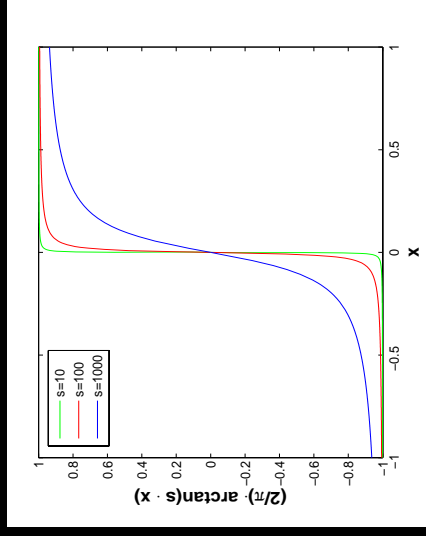
- **Why nonsmooth?**
 - There is an established theory for smooth dynamical systems (analysis, bifurcations and control).
 - Many systems are nonsmooth on a macroscopic scale.
 - Think for example of systems with switches, friction, impacts, walking mechanism, etc.
 - There is no consistent theory for those cases where the vector field is nonsmooth and/or non-differentiable.
 - Then what?

Introduction and motivation

■ ... Better smooth?

➤ Typically, nonsmooth characteristic can be smoothed out, e.g.

- Well, this could work but:
 - ✓ Usually, eqns become stiff.
 - ✓ Time- consuming to simulate.
 - ✓ Solution might not be accurate

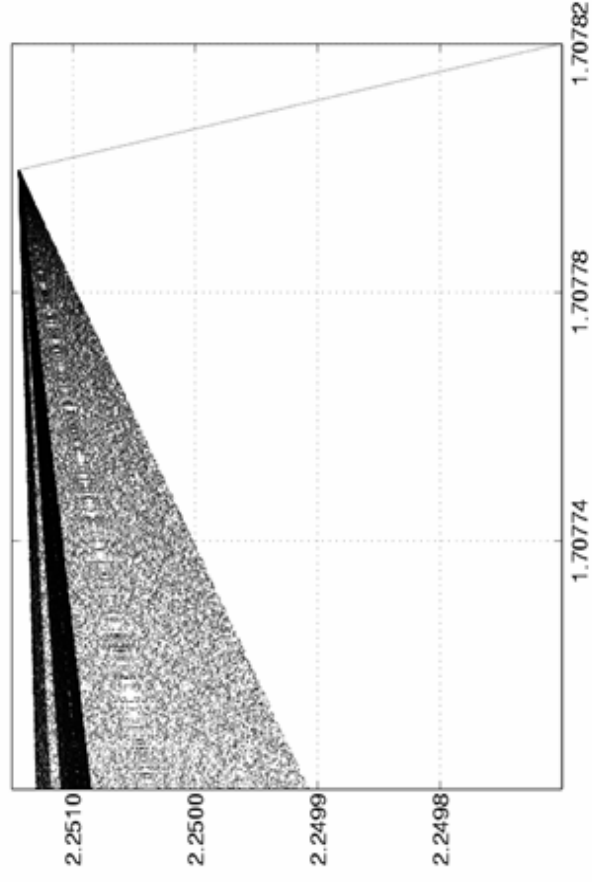


➤ Take, for example, a block sliding on an inclined plane...

Introduction and motivation

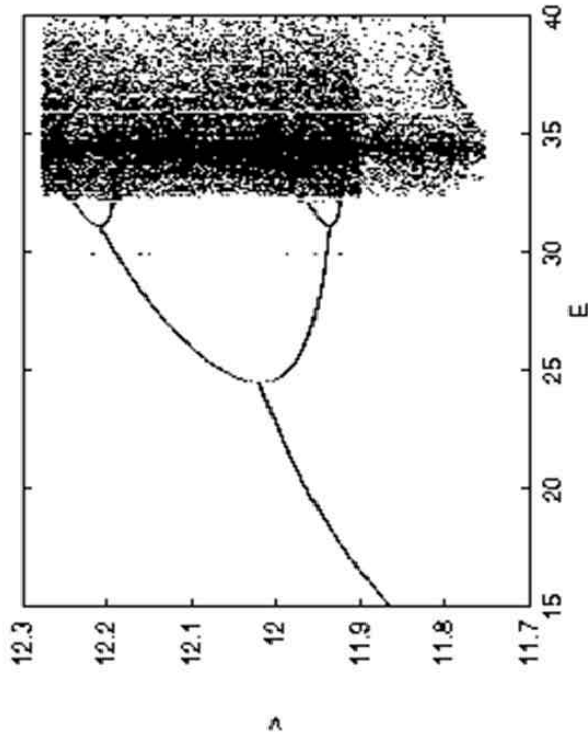
- **Smoothing:**
 - So, smoothing can be an option but it must be dealt with great care.
 - Several techniques can be used but no detailed study/comparison of the techniques has been carried out so far.
 - Maybe dealing with the nonsmooth equations can be easier.
 - But to do that we need a consistent theory of nonsmooth systems... (Complementarity systems, Hybrid systems,...)

Introduction and motivation



Friction oscillators

DC-DC buck converter



Background and literature review

- **Classification of nonsmooth dynamical systems.**
- **Modelling using complementarity formalism.**
- **Classification of simulation techniques:**
 - **Event-driven methods.**
 - **Smoothing methods.**
 - **Time-stepping methods.**
- **Bifurcation analysis.**

Classification of nonsmooth dynamical systems

NON-SMOOTH DYNAMICAL SYSTEMS

SYSTEMS WITH A NONSMOOTH
CONTINUOUS VECTOR FIELD

Example:

MECHANICAL SYSTEMS WITH
BI-LINEAR ELASTIC SUPPORT

FILIPPOV SYSTEMS

Example:

POWER ELECTRONIC CONVERTERS
and DRY FRICTION OSCILLATORS

SYSTEMS WITH JUMPS
IN THE STATE

Example:

IMPACTING SYSTEMS and
VIBRO-IMPACTING MACHINES

Ref.: BIFURCATIONS IN NONLINEAR DISCONTINUOUS SYSTEMS. "R.I.LEINE, D.H.VAN CAMPEN
and B.L.VAN DE VRANDE".

Modelling using complementarity formalism

■ Linear Complementarity Problem (LCP):

Given a **matrix** $M \in R^{k \times k}$ and a **vector** $q \in R^k$. The **linear complementarity problem** $LCP(q,M)$ is to find vectors $u, y \in R^k$ such that

$$y = q + M \cdot u,$$

and satisfying

$$u_i \geq 0, y_i \geq 0, \text{ and } \{u_i = 0 \text{ or } y_i = 0\}, \forall i \in \{1, \dots, k\}$$

or to show that no such vector u exists.

The conditions $u \geq 0, y \geq 0, u^T \cdot y = 0$ are called **complementarity conditions (CC)** and are denoted by $0 \leq u \perp y \geq 0$.

Modelling using complementarity formalism

Theorem 1: If $M \in R^{k \times k}$ is **positive definite**, then the **LCP** (q, M) has a **unique solution** for all $q \in R^k$.

In general, the LCP with a **positive semi-definite matrix** can have **multiple solutions**. For instance, the LCP with:

$$q = \begin{pmatrix} -1 \\ -1 \end{pmatrix} \quad M = \begin{pmatrix} 1 & 1 \\ 1 & 1 \end{pmatrix}$$

has solutions

$$u_1 = (1, 0), \quad u_2 = (0, 1), \quad u_3 = \left(\frac{1}{2}, \frac{1}{2}\right)$$

Modelling using complementarity formalism

Theorem 1: If $M \in R^{k \times k}$ is **positive definite**, then the **LCP (q, M)** has a **unique solution** for all $q \in R^k$.

Definition 1: A matrix $M \in R^{k \times k}$ is said a **P-matrix** if all its **principal minors** are **positive**. The class of such matrices is denoted P.

Theorem 2: A matrix $M \in R^{k \times k}$ is a **P-matrix** if and only if the **LCP (q, M)** has a **unique solution** for all $q \in R^k$.

Remark: Several pivoting methods have been devoted to solve numerically LCPs. Some examples are Lemke's and Murty's algorithms

Modelling using complementarity formalism

■ Linear Complementarity Systems (LCS).

Basically, a **linear complementarity system (LCS)** is a **combination of a standard linear system and complementarity conditions.**

Therefore, a complementarity system is given by:

$$\dot{x}(t) = A \cdot x(t) + B \cdot u(t) + E$$

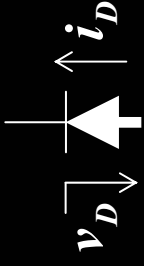
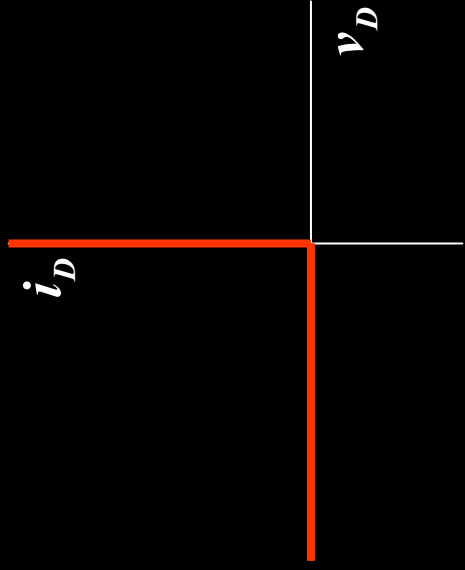
$$y(t) = C \cdot x(t) + D \cdot u(t) + F$$

$$0 \leq u(t) \perp y(t) \geq 0, \quad \forall t \in I.$$

Remark: If the positiveness in the complementarity conditions (CC) is relaxed, i.e. the complementarity variables are orthogonal but the positiveness are not required we obtain a **LINEAR CONE COMPLEMENTARITY SYSTEM (LCCS).**

Modelling using complementarity formalism

Characteristic curve of an ideal diode



$$0 \leq -v_D \perp i_D \geq 0$$

In the context of electrical circuits, imposing complementarity conditions simply means that some ports are terminated by ideal diodes, with the current i_D and (minus) the voltage v_D as complementarity variables.

Modelling using complementarity formalism

■ **Why Complementarity Systems?**

- Complementarity systems are particularly suited to describe systems with unilateral constraints (diodes, impact oscillators, friction, saturations, relays, VSS).
- Systems with a large number of contacts, switches,...
- Routines from optimization can be used.
- The formalism is compact while retaining its physical meaning...

Modelling using complementarity formalism

- **Thesis results in this chapter:**
 - We have modelled some basic dc-dc power converters (buck, boost, buck-boost, Ćuk).
 - After fixing the position of the switches, the system incorporates, in a natural way, the description of generalised discontinuous conduction mode (GDCM), characterised by a reduction of the dimension of the system.
 - Analytical state-space conditions have been stated for the presence of GDCM.
 - Modelling, analysis and simulation of a parallel resonant converter (PRC).
 - Simulation of a boost power converter with sliding mode control, even though a general control theory for complementarity systems is not still developed.

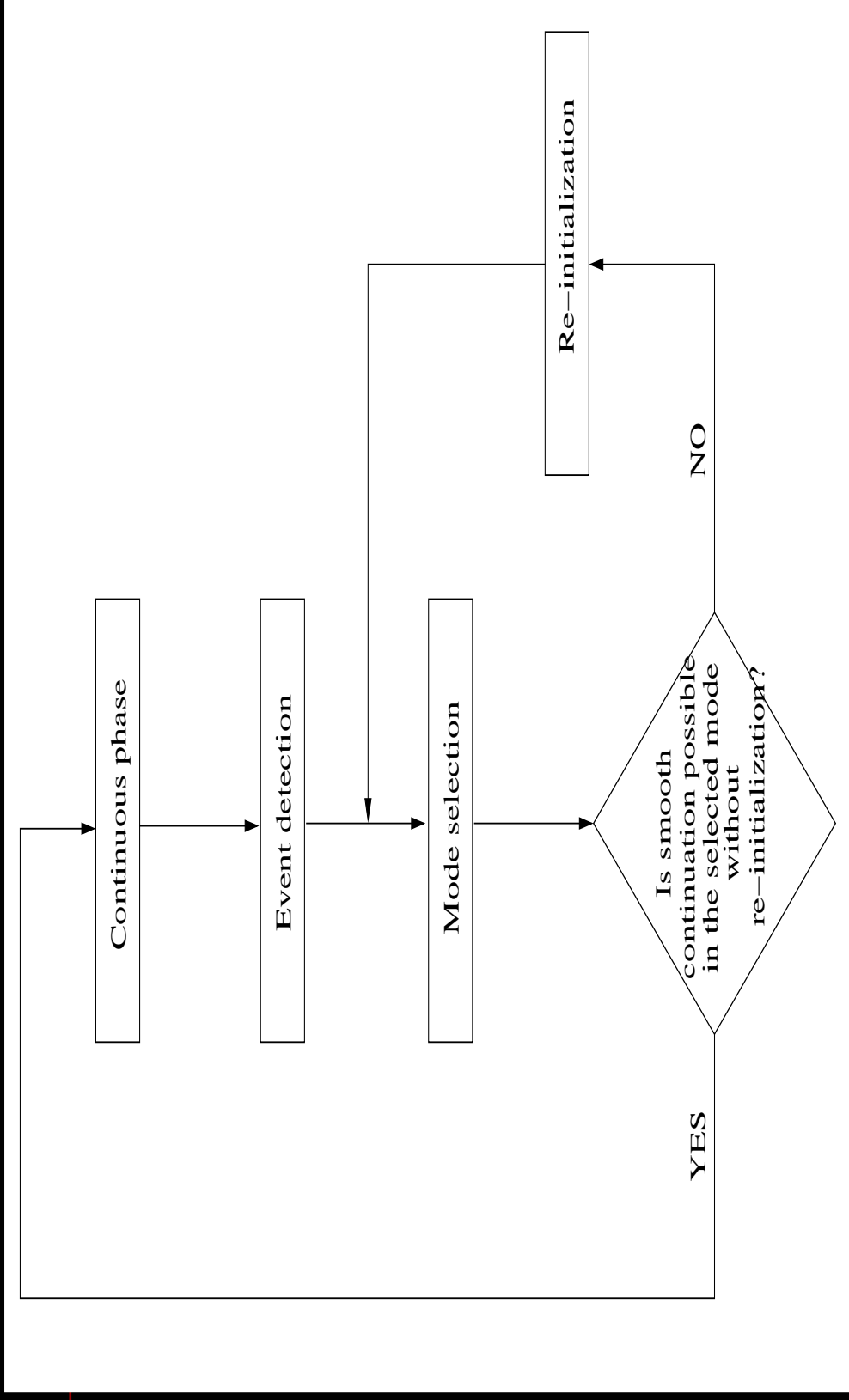
Classification of simulation techniques

Moreau classifies the literature on simulation techniques for rigid body dynamics with collisions into three categories:

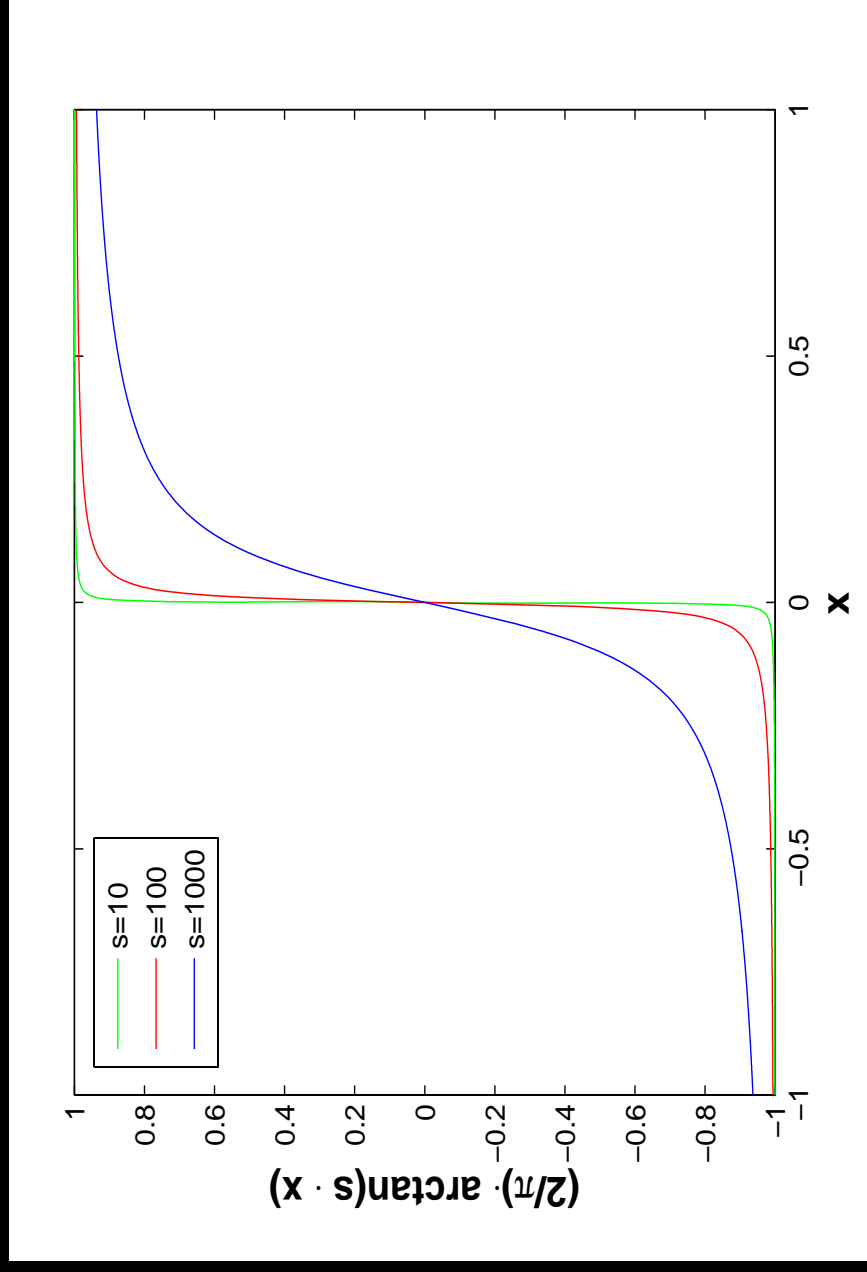
- **Event-driven methods.**
- **Smoothing methods.**
- **Time-stepping methods.**

Heemels in his thesis shows that this classification also applies to possible numerical methods for complementarity systems.

Event-driven method



Smoothing method



Smoothing method

- **Discrete mode do not really exist**, so event detection and mode selection are not necessary.
- Instantaneous jumps are replaced by (finitely) **fast motions**, so also the problem of **re-initialization disappears**.
- **Drawback**: An accurate simulation requires the use of **very stiff** approximate laws. Then, a **very small step-length** is needed and possibly **artificial terms** in the equations are required in order to enforce numerical stability.

Time-stepping method

- Time-stepping methods replace the describing equations directly by some “discretized” equivalent.
- At each time-step an algebraic problem (called “one-step problem”) involving information obtained from previous time-steps is solved.

$$\frac{x_{k+1}^h - x_k^h}{h} = A \cdot x_{k+1}^h + B \cdot u_{k+1}^h + E$$

$$y_{k+1}^h = C \cdot x_{k+1}^h + D \cdot u_{k+1}^h + F$$

$$0 \leq y_{k+1}^h \perp u_{k+1}^h \geq 0$$

Backward Euler Time-stepping method

Algorithm 1: $(\{u_{k+1}^h\}, \{x_{k+1}^h\}, \{y_{k+1}^h\}) = \text{LCPsimulator}(A, B, C, D, T_{\text{end}}, h, x_0)$

$$1. N_h = \left\lfloor \frac{T_{\text{end}}}{h} \right\rfloor.$$

$$2. x_{-1}^h := x_0.$$

$$3. k := 1.$$

4. *solve the one-step problem*

$$y_{k+1}^h = C \cdot (\text{Id} - h \cdot A)^{-1} \cdot x_k^h + [D + h \cdot C \cdot (\text{Id} - h \cdot A)^{-1} \cdot B] u_{k+1}^h + h \cdot (\text{Id} - h \cdot A)^{-1} \cdot E + F$$

$$0 \leq u_{k+1}^h \perp y_{k+1}^h \geq 0$$

$$5. x_{k+1}^h := (\text{Id} - h \cdot A)^{-1} \cdot x_k^h + h \cdot (\text{Id} - h \cdot A)^{-1} \cdot (B \cdot u_{k+1}^h + E)$$

$$6. k := k + 1.$$

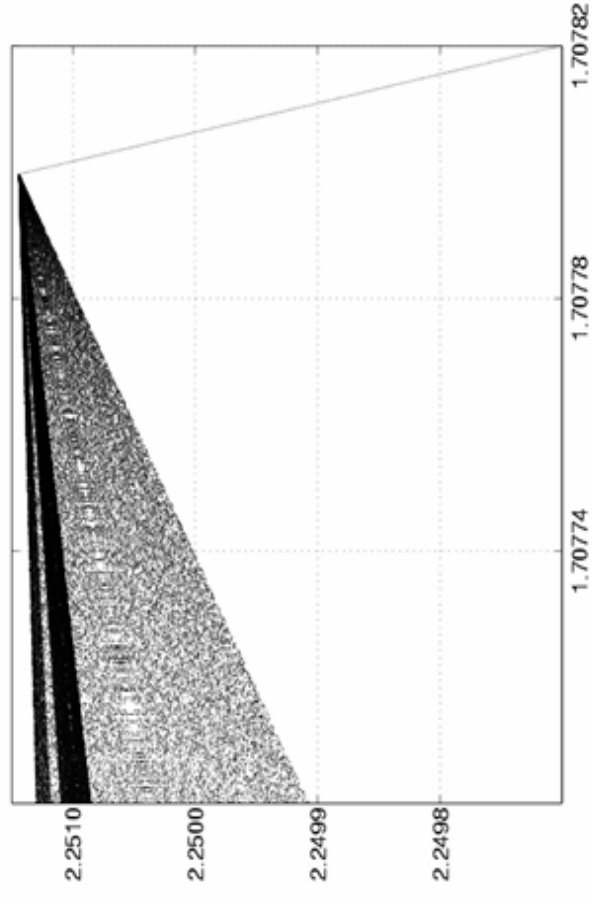
7. *if* $k < N_h$ *go to* 4.

8. *stop.*

Time-stepping method

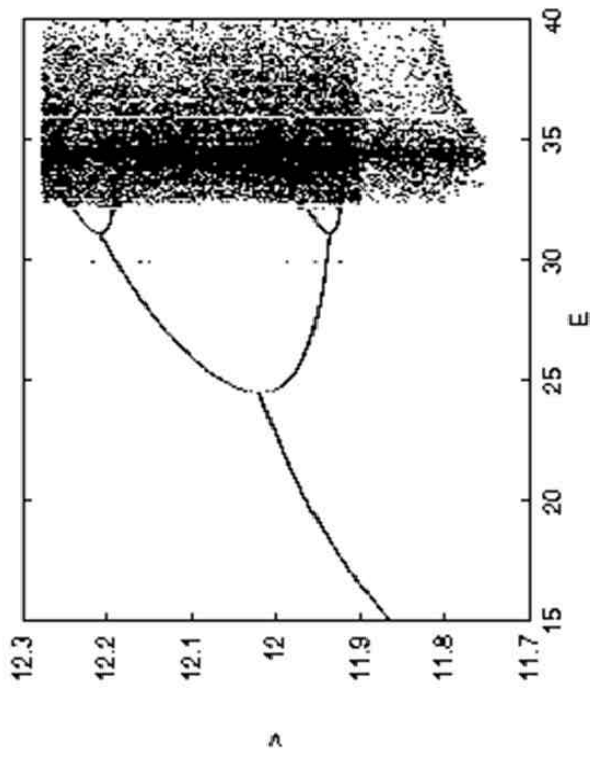
- Time-stepping methods do not determine the event times accurately, but “overstep” them, which puts the consistency of the method into question.
- But, if consistency is proved it is a efficient method for simulation of NSDS with a large number of contacts, diodes,...
- For linear relay systems the Backward Euler method is consistent.
- Drawback: It's a low order integration method.

Bifurcation analysis



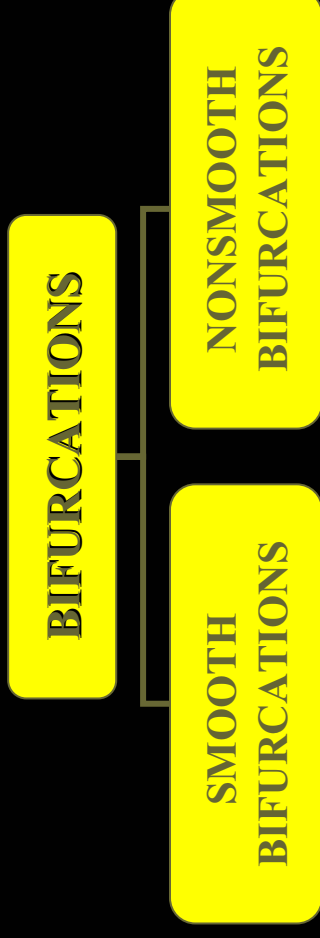
Friction oscillators

DC-DC buck converter



Bifurcation analysis

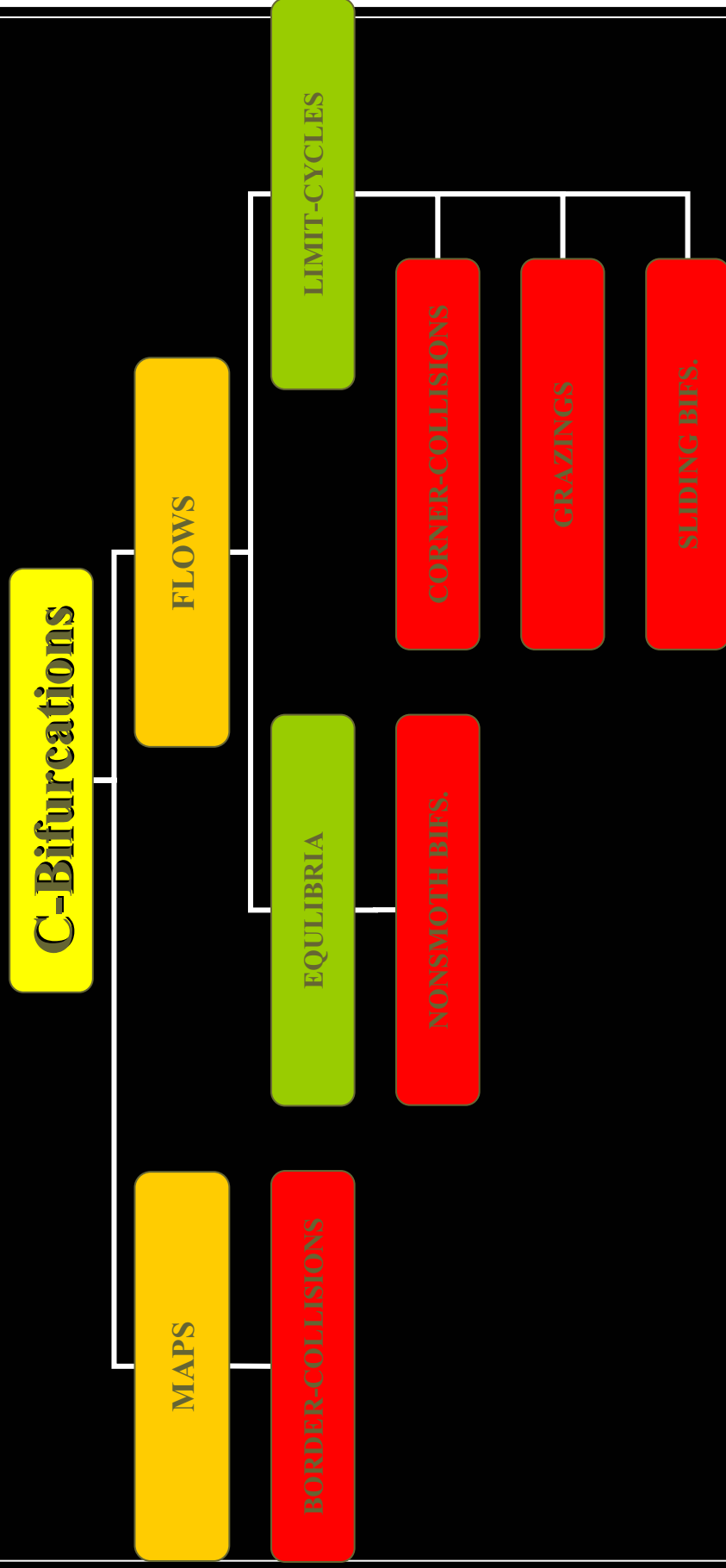
- Codimension-one nonsmooth bifurcations:



Definition:

The appearance of a topologically non-equivalent phase portrait under the variation of a parameter is called a **bifurcation**.

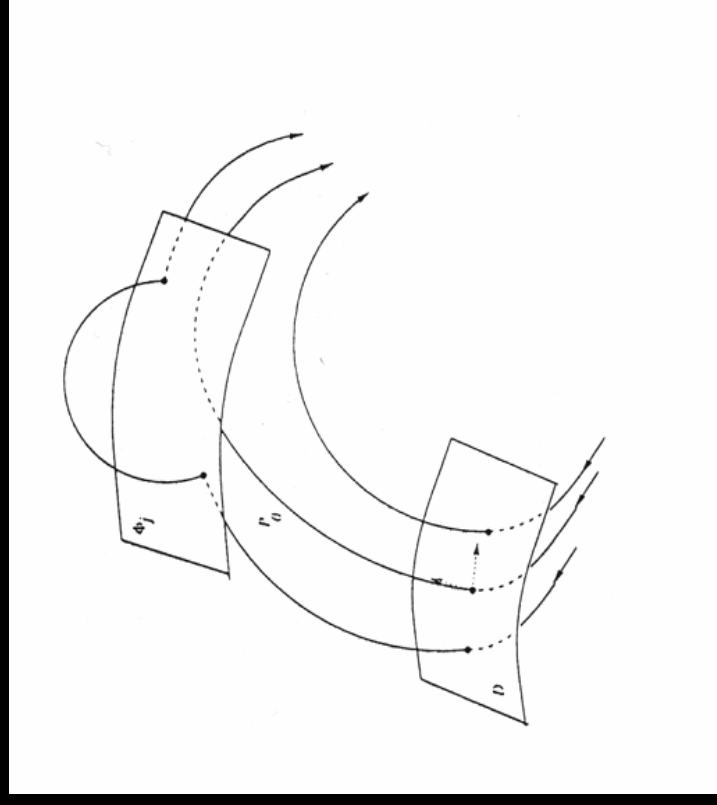
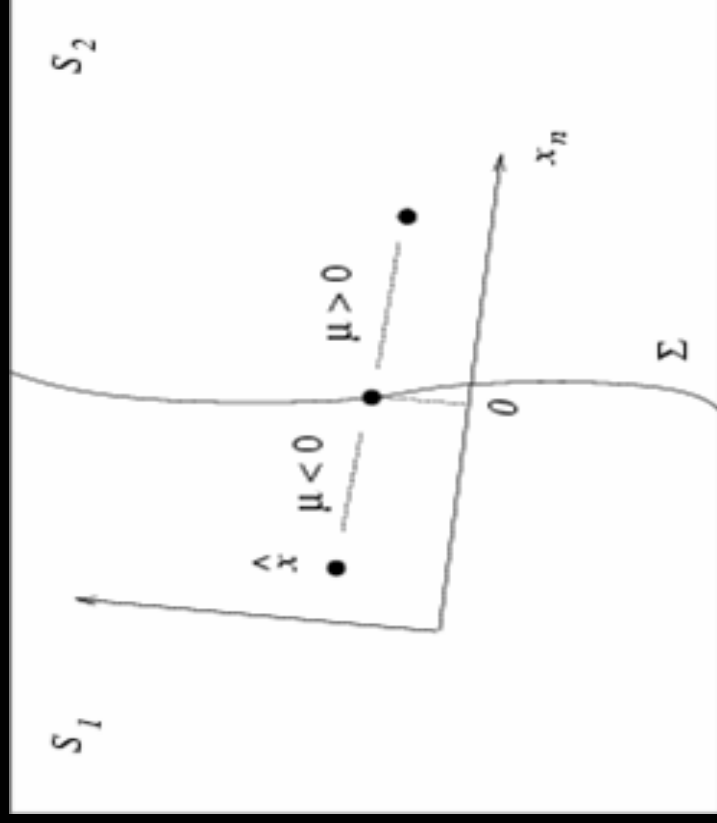
Bifurcation analysis



Bifurcation analysis

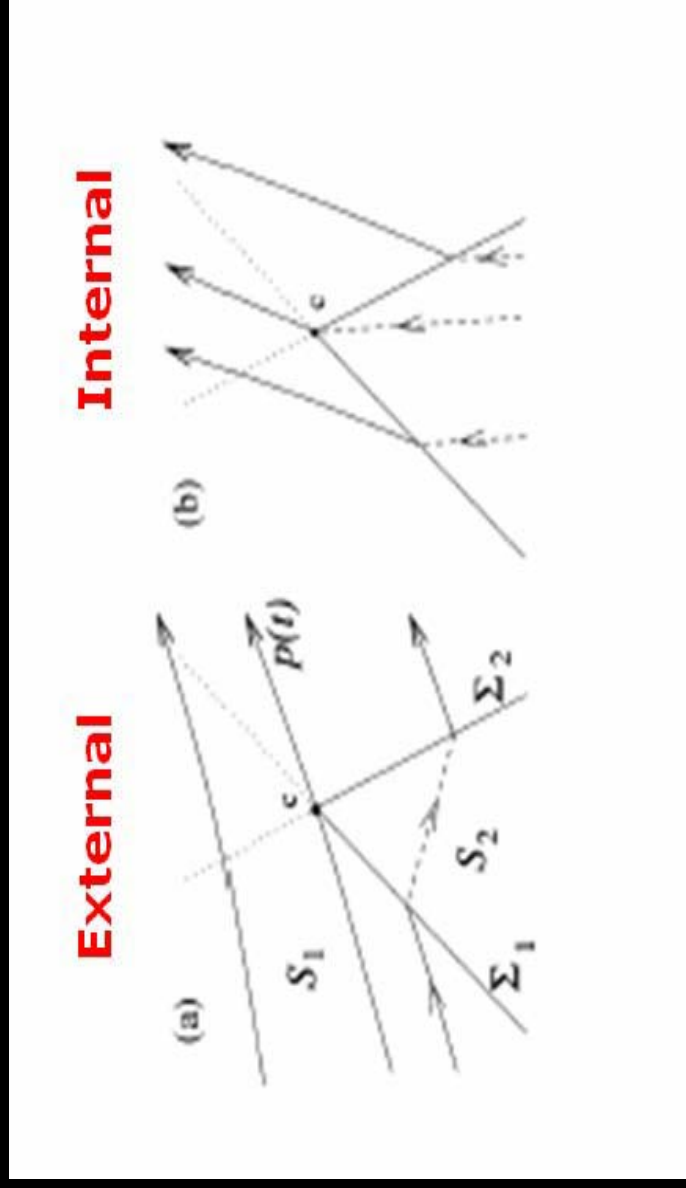
Border-Collision

Grazing



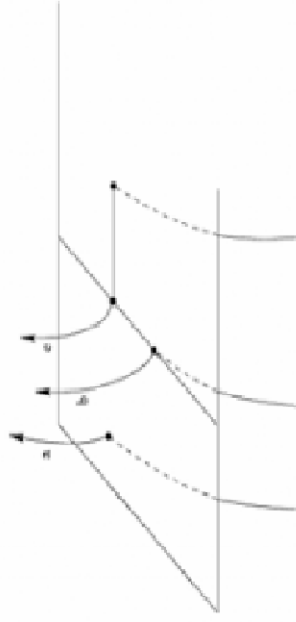
Bifurcation analysis

- We called a **Corner-Collision bifurcation** if the boundary is itself non-smooth

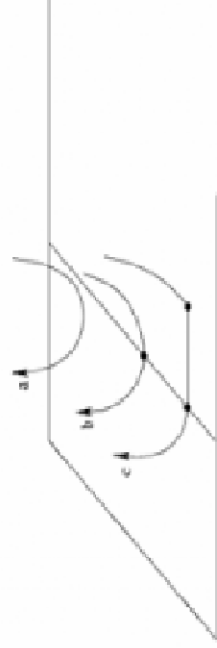
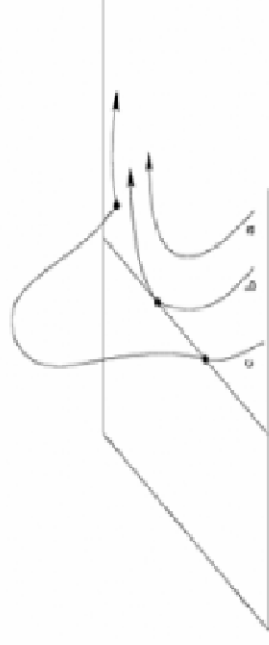


Bifurcation analysis

Crossing-sliding

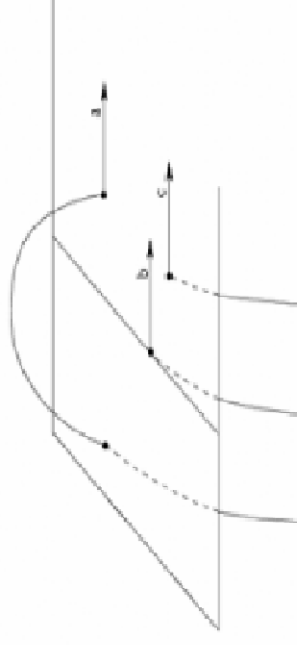


Adding-sliding

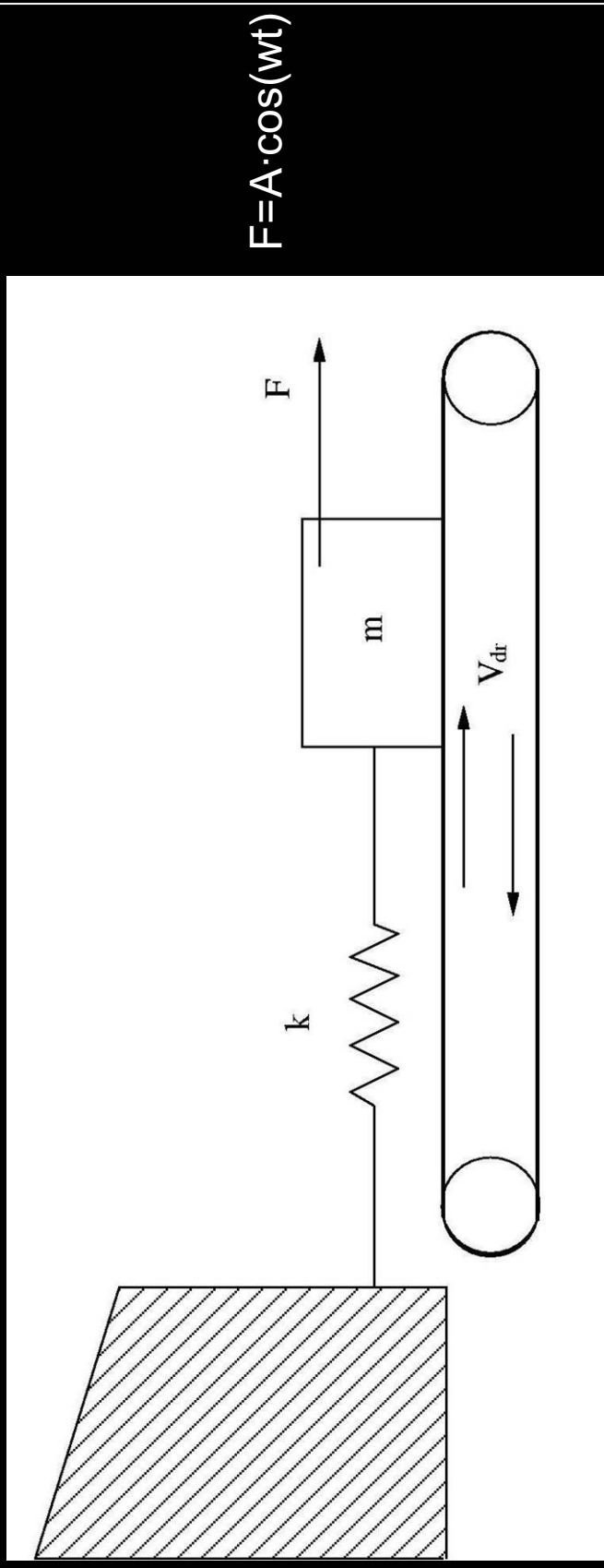


Grazing-sliding

Switching-sliding

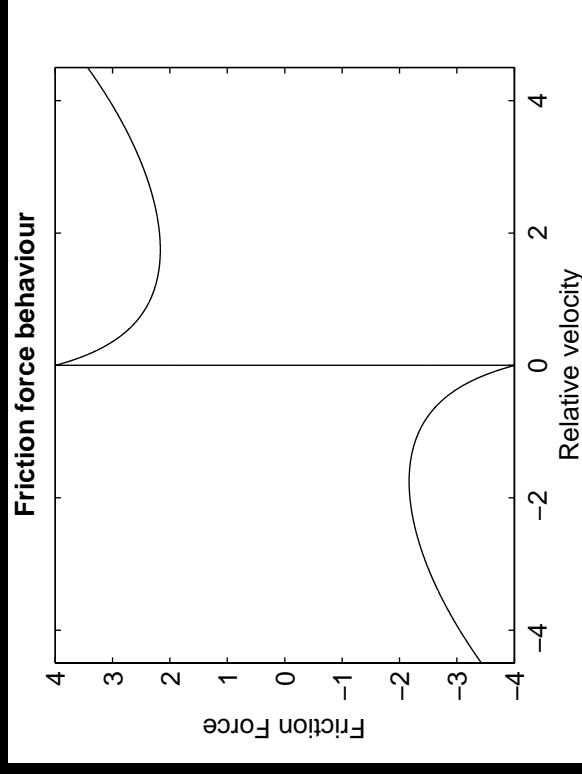


Sliding Bifurcations in a dry friction oscillator



Description of the model

- Approximation of a measured friction characteristic introduced by Popp:



$$F_{fr,k} = mg\mu(v_{rel})$$

$$\mu(v_{rel}) = \frac{\alpha}{1 + \gamma \cdot |v_{rel}|} + \beta + \eta \cdot v_{rel}^2$$

$$\alpha = 0.3, \gamma = 1.42, \beta = 0.1, \eta = 0.01$$

Filippov system

We consider a discontinuous surface Σ , which is defined by a smooth scalar function $h(x)$:

$$\Sigma = \{x \in \mathbb{R}^3 : h(x) = x_2 - V_{dr} = 0\}$$

Then, we can formulate our system as a Filippov system:

$$\dot{x} = \begin{cases} F_1(x, \omega) & \text{if } h(x) < 0 \\ F_2(x, \omega) & \text{if } h(x) > 0 \end{cases}$$

$$F_1 = \begin{cases} x_2 \\ -\frac{k}{m}x_1 + \frac{A}{m}\cos(x_3) + \frac{F_{fr,k}}{m} \\ \omega \end{cases}; \quad F_2 = \begin{cases} x_2 \\ -\frac{k}{m}x_1 + \frac{A}{m}\cos(x_3) - \frac{F_{fr,k}}{m} \\ \omega \end{cases}$$

Filippov system

Now, we can define the sliding region as

$$\hat{\Sigma} = \left\{ x \in \Sigma : \left| \frac{-kx_I + A \cos(\omega t)}{F_{fr,s}} \right| < 1 \right\}$$

with boundaries:

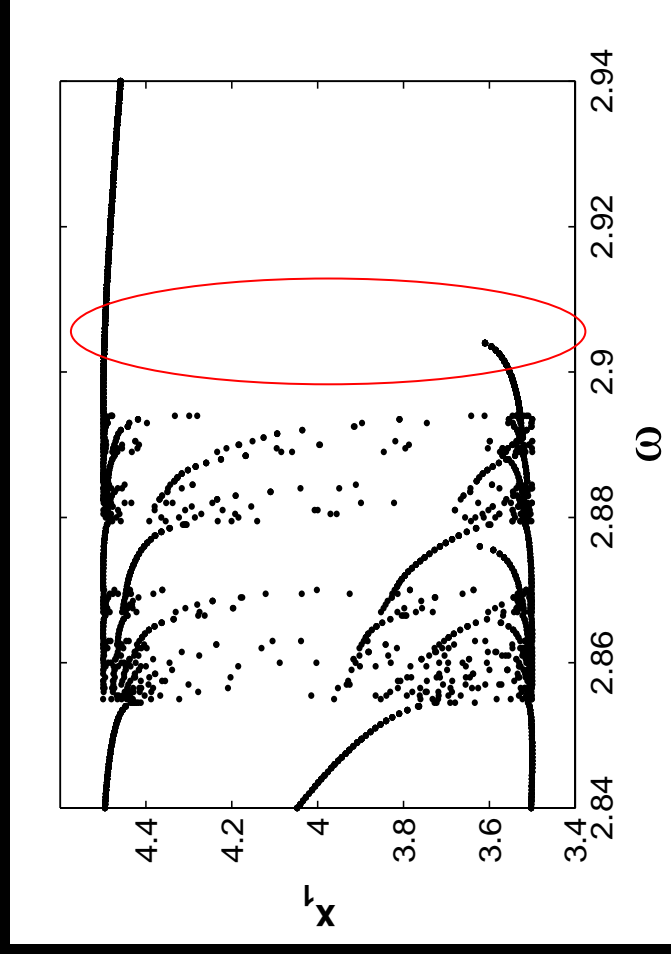
$$\partial\Sigma^+ = \left\{ x \in \Sigma : x_I = \frac{-F_{fr,s}}{k} + \frac{A}{k} \cos(\omega t) \right\}$$

$$\partial\Sigma^- = \left\{ x \in \Sigma : x_I = \frac{F_{fr,s}}{k} + \frac{A}{k} \cos(\omega t) \right\}$$

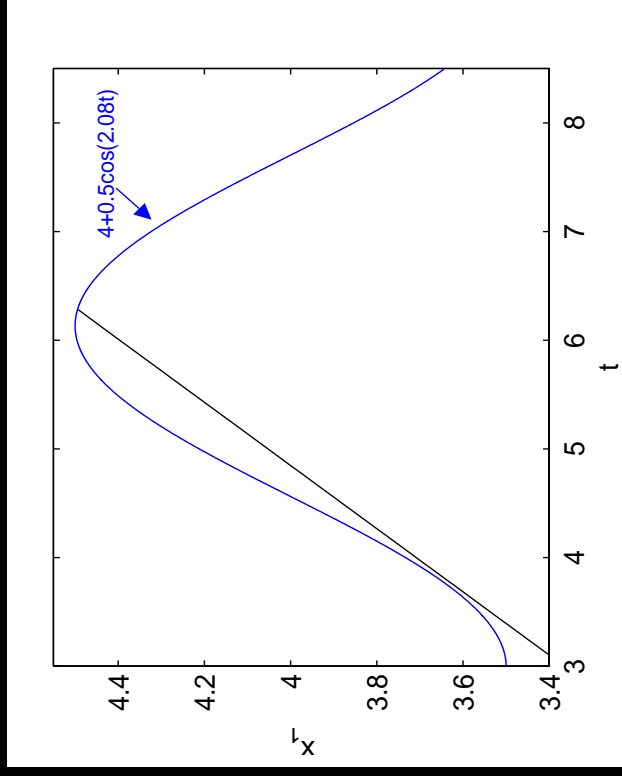
Parameters:

$$\begin{aligned} k &= 1, \\ m &= 1, \\ g &= 10, \\ V_{dr} &= 1. \end{aligned}$$

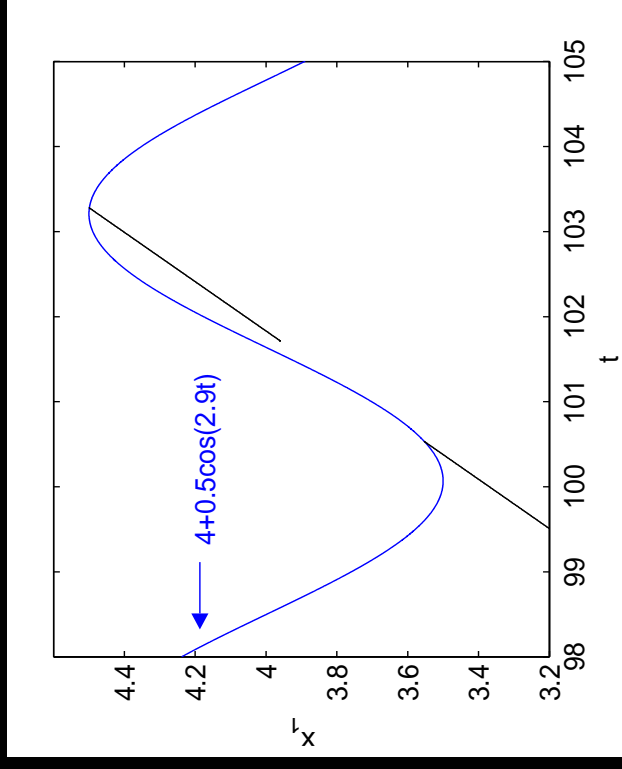
Simulations



Adding-sliding Bifurcation

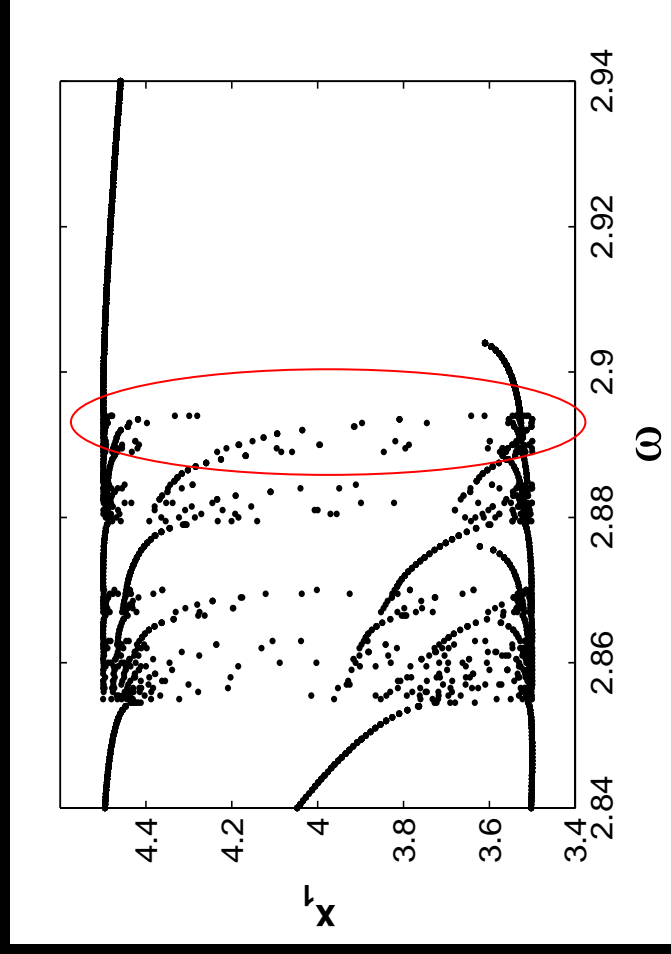


$W = 2.908$



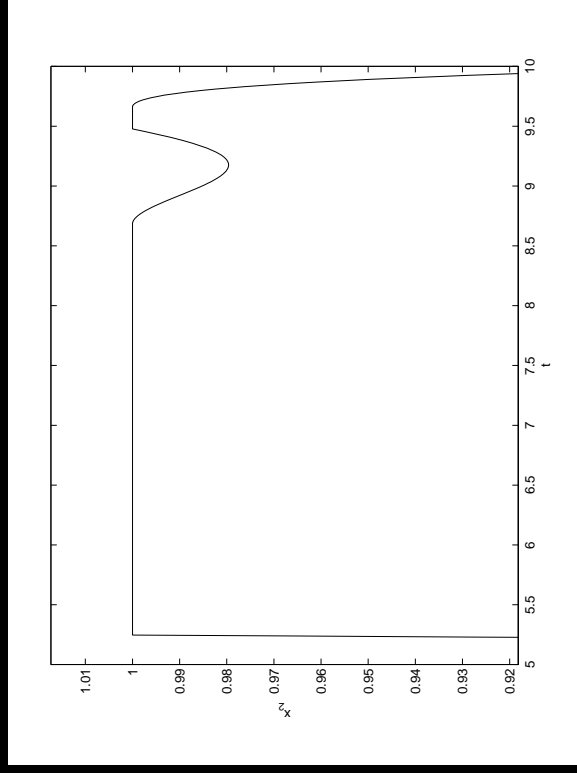
$W = 2.9$

Simulations

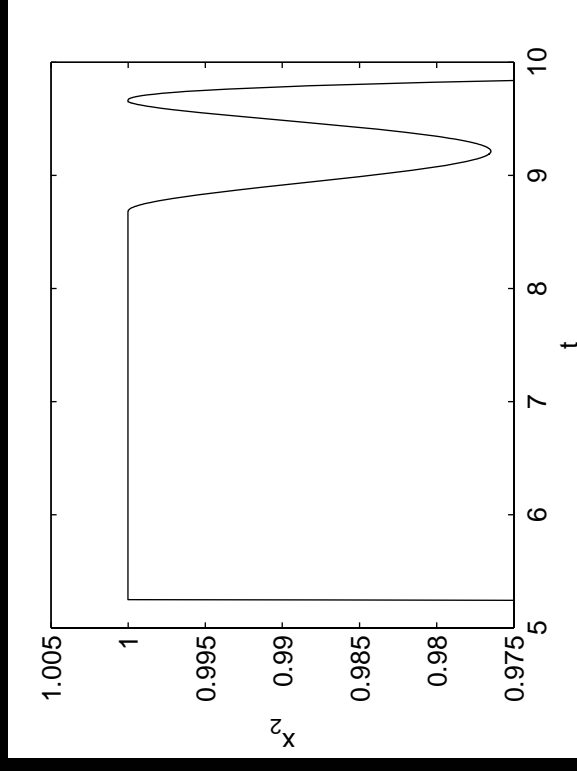


Grazing-sliding Bifurcation

Bifurcation at $(A, w) = (0.5, 2.8942)$



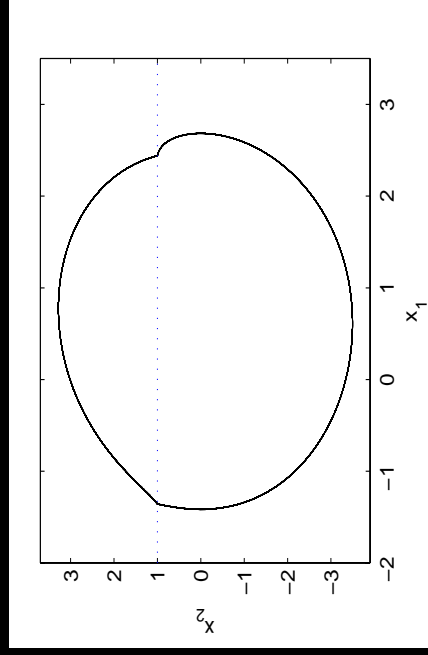
$W = 2.895$



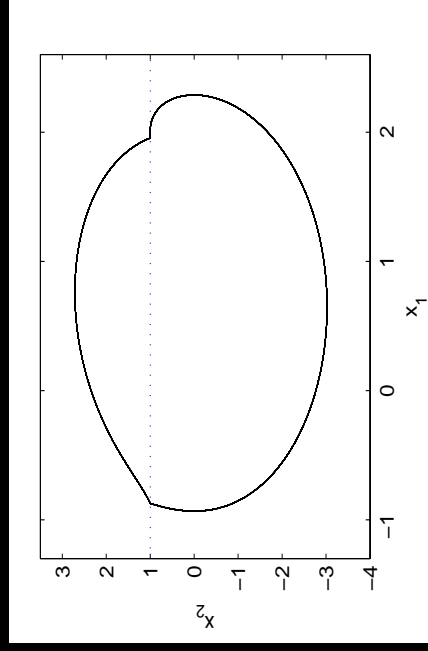
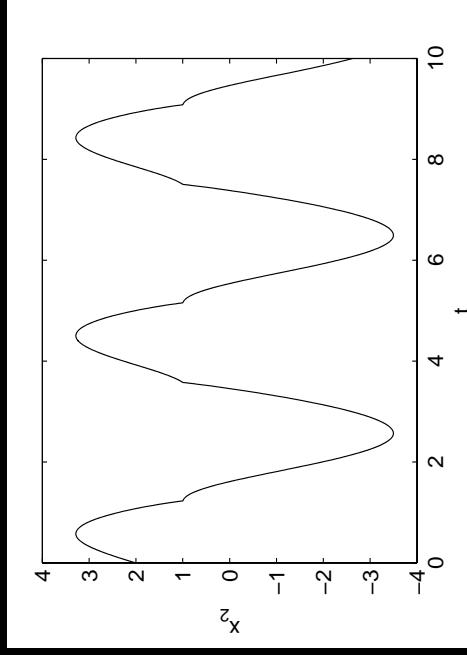
$W = 2.893$

Crossing-sliding Bifurcation

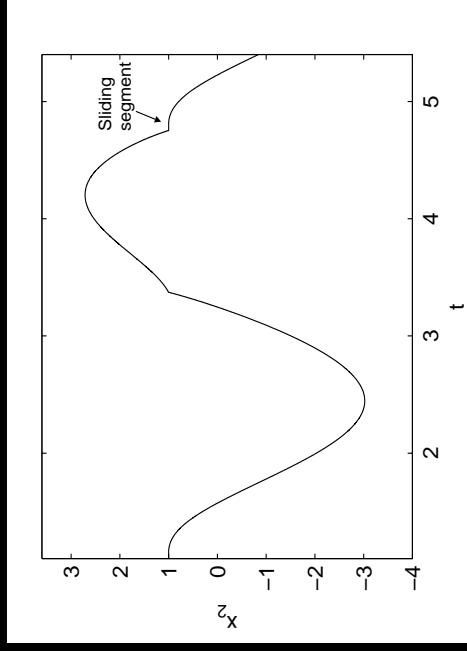
Bifurcation at $(A, w) = (5, 1.65768)$



$W = 1.6$

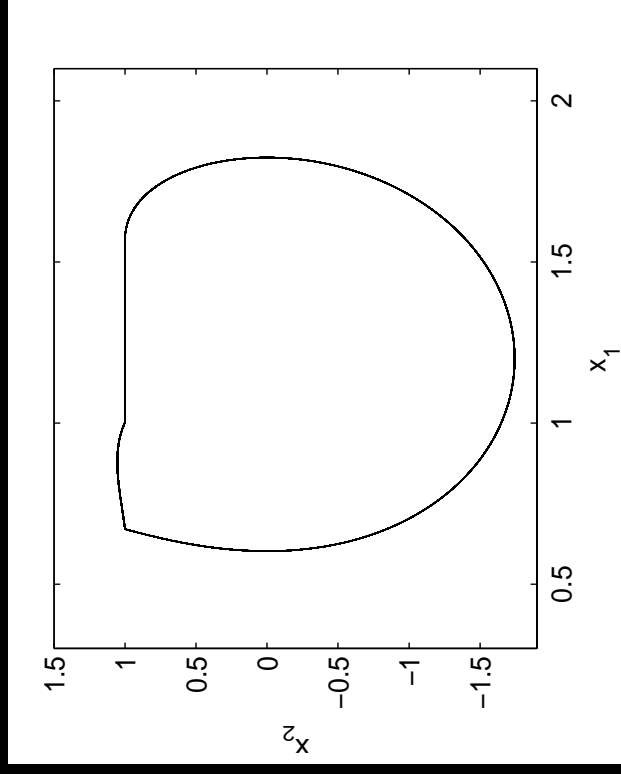


$W = 1.72$

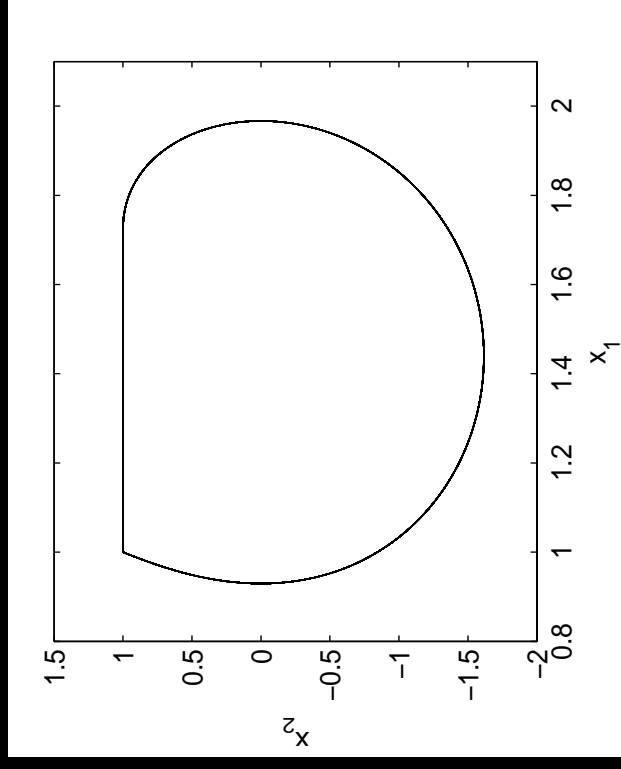


Switching-sliding Bifurcation

Bifurcation at $(A, w) = (5, 2.7992)$

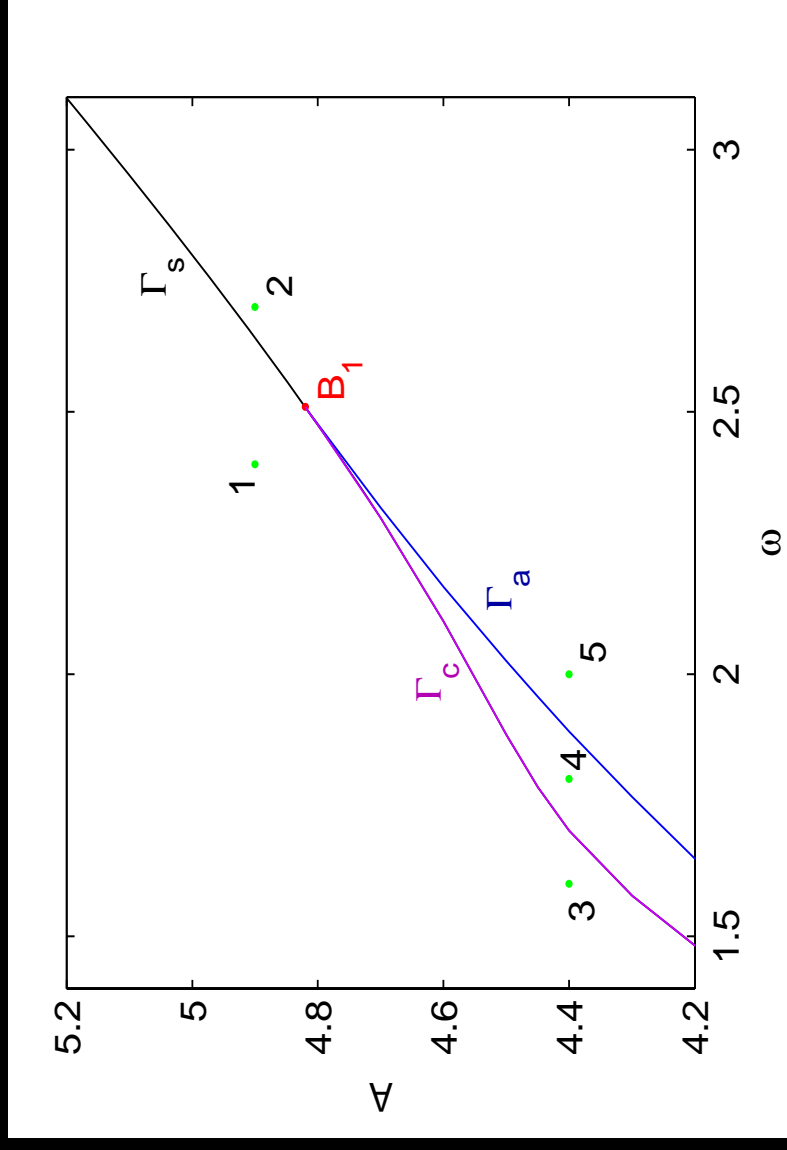


$W = 2.5$



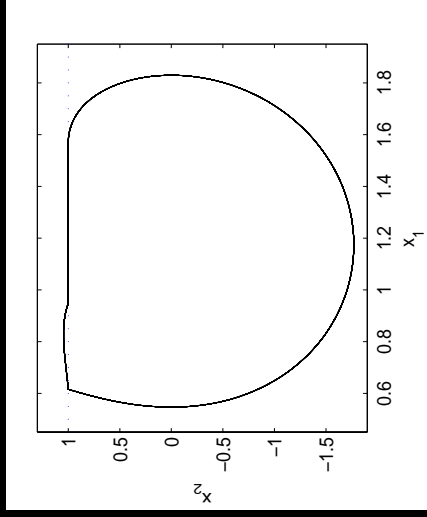
$W = 2.8$

Degenerate Switching-sliding Bifurcation

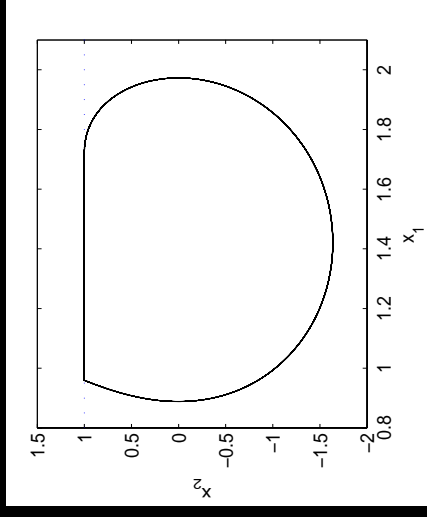


Degenerate Switching-sliding Bifurcation

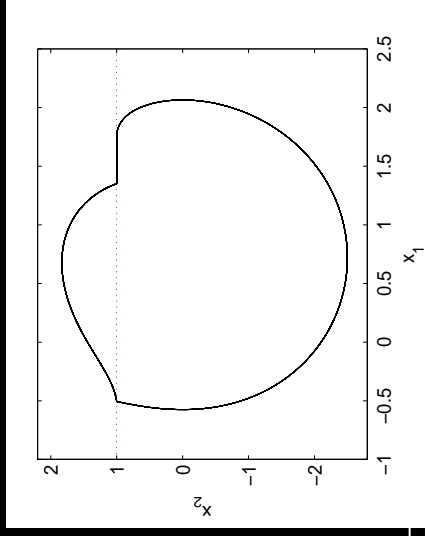
Point 1



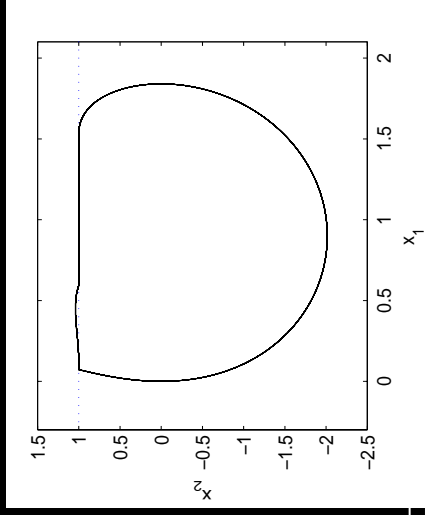
Point 2



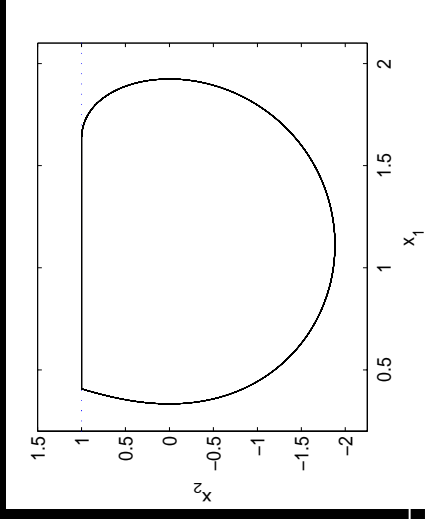
Point 3



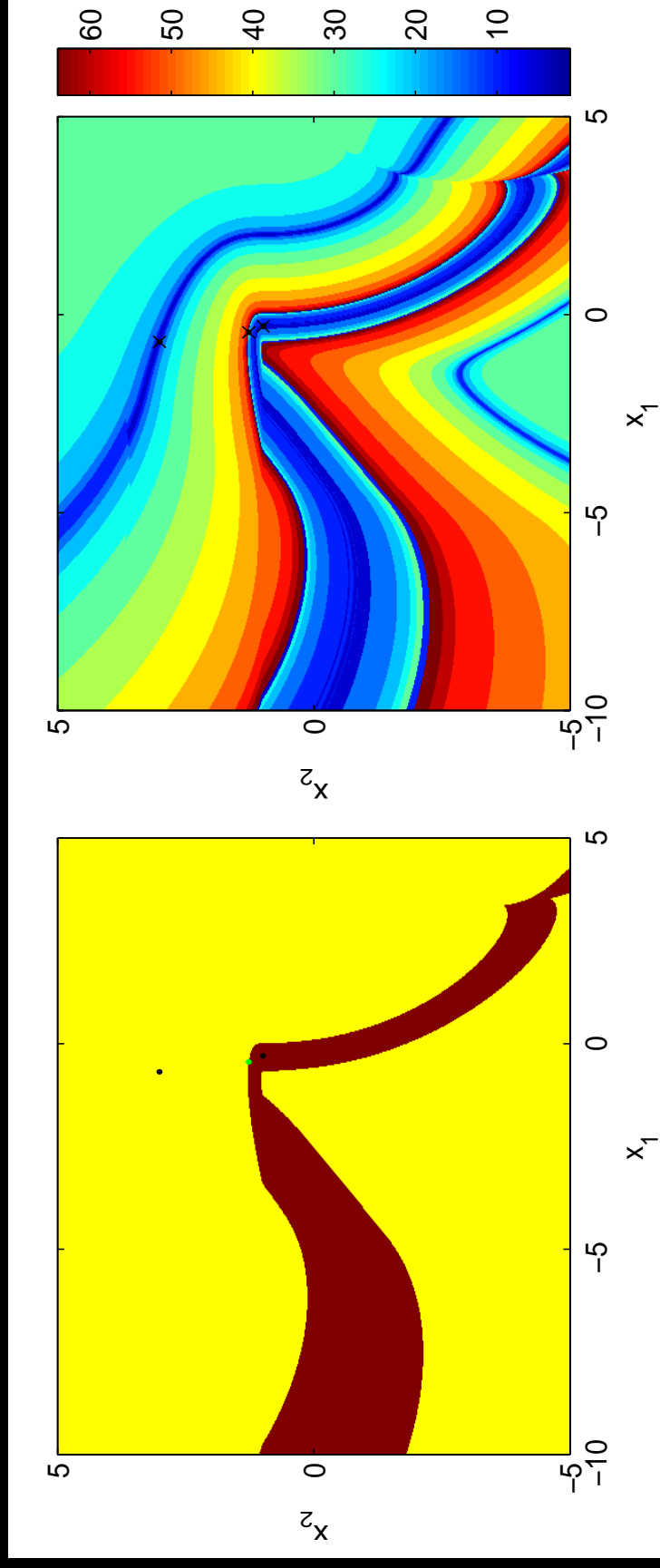
Point 4



Point 5

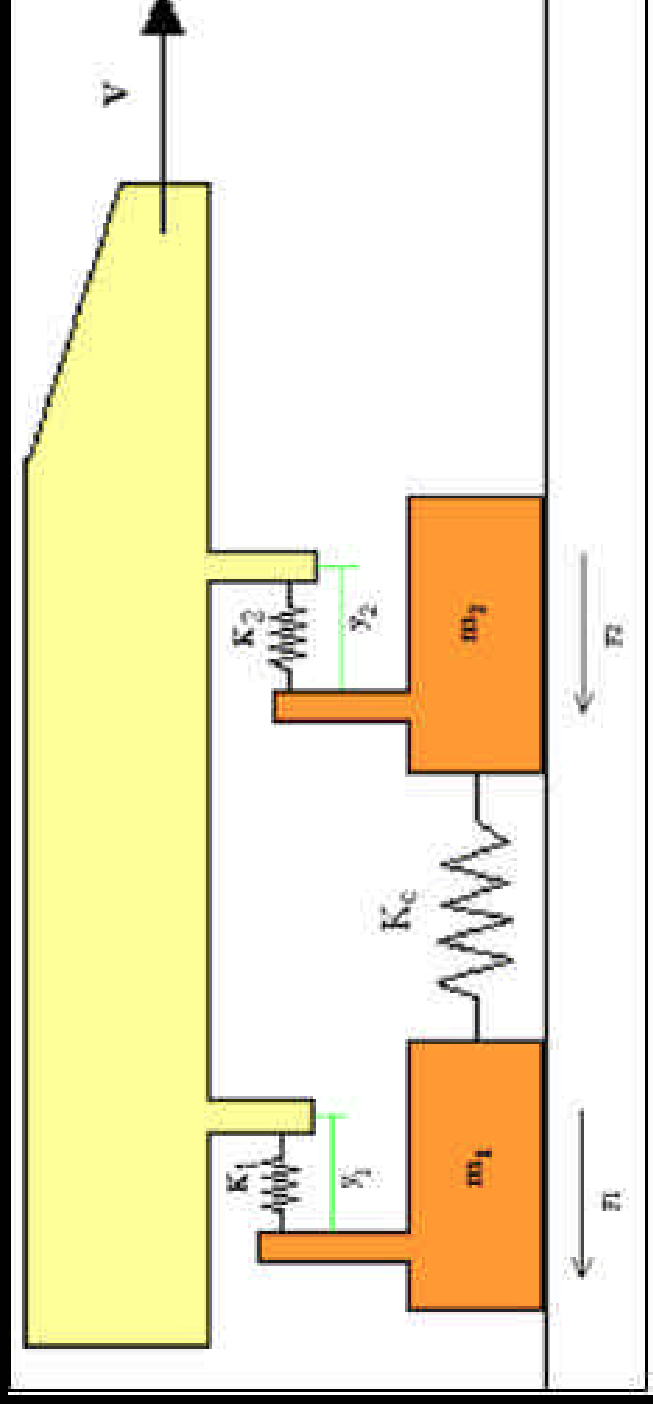


Domain of Attraction

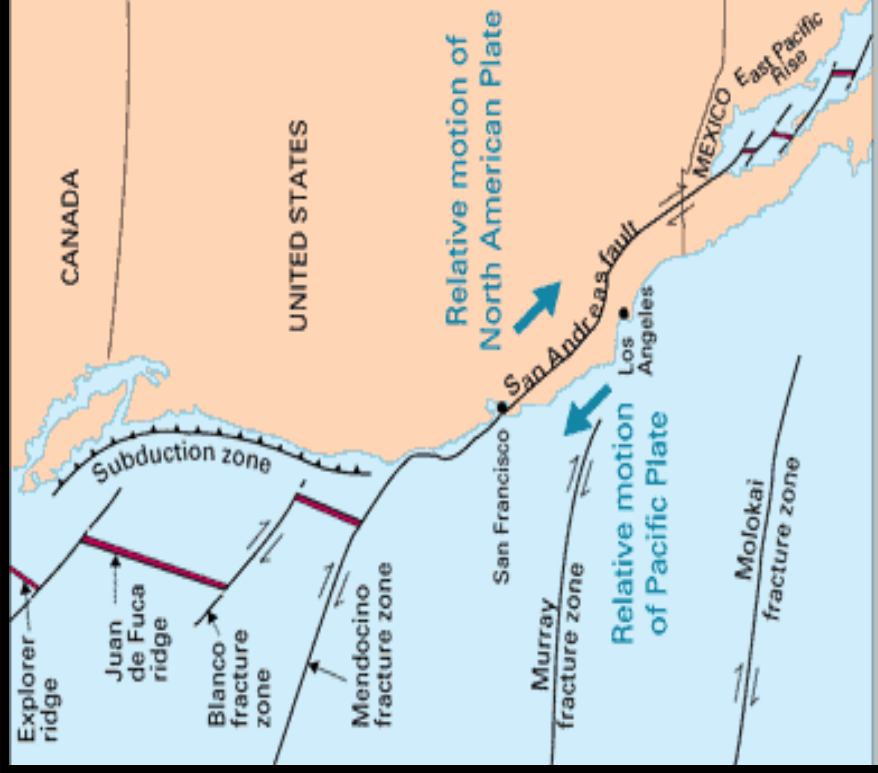


$A = 3.6$ $W = 1.067$

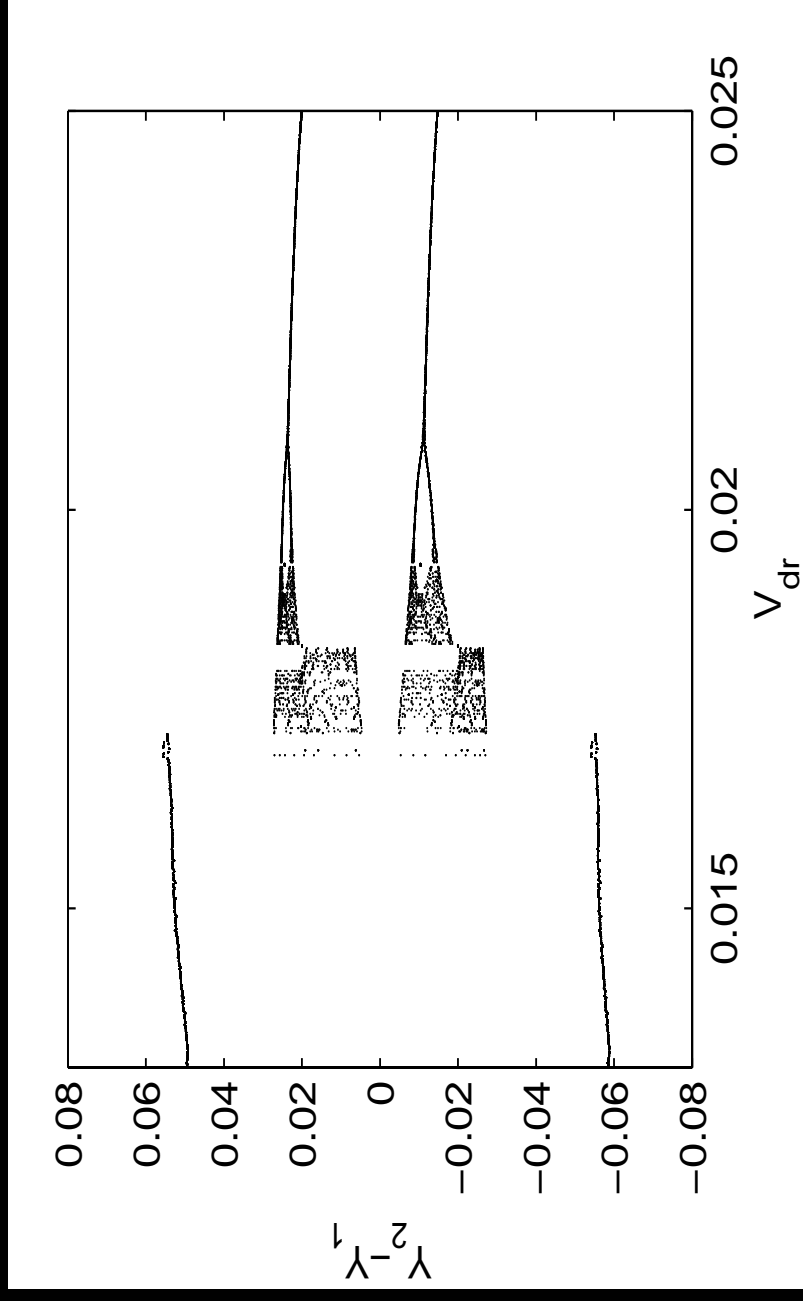
Bifurcations in a two-block stick-slip system



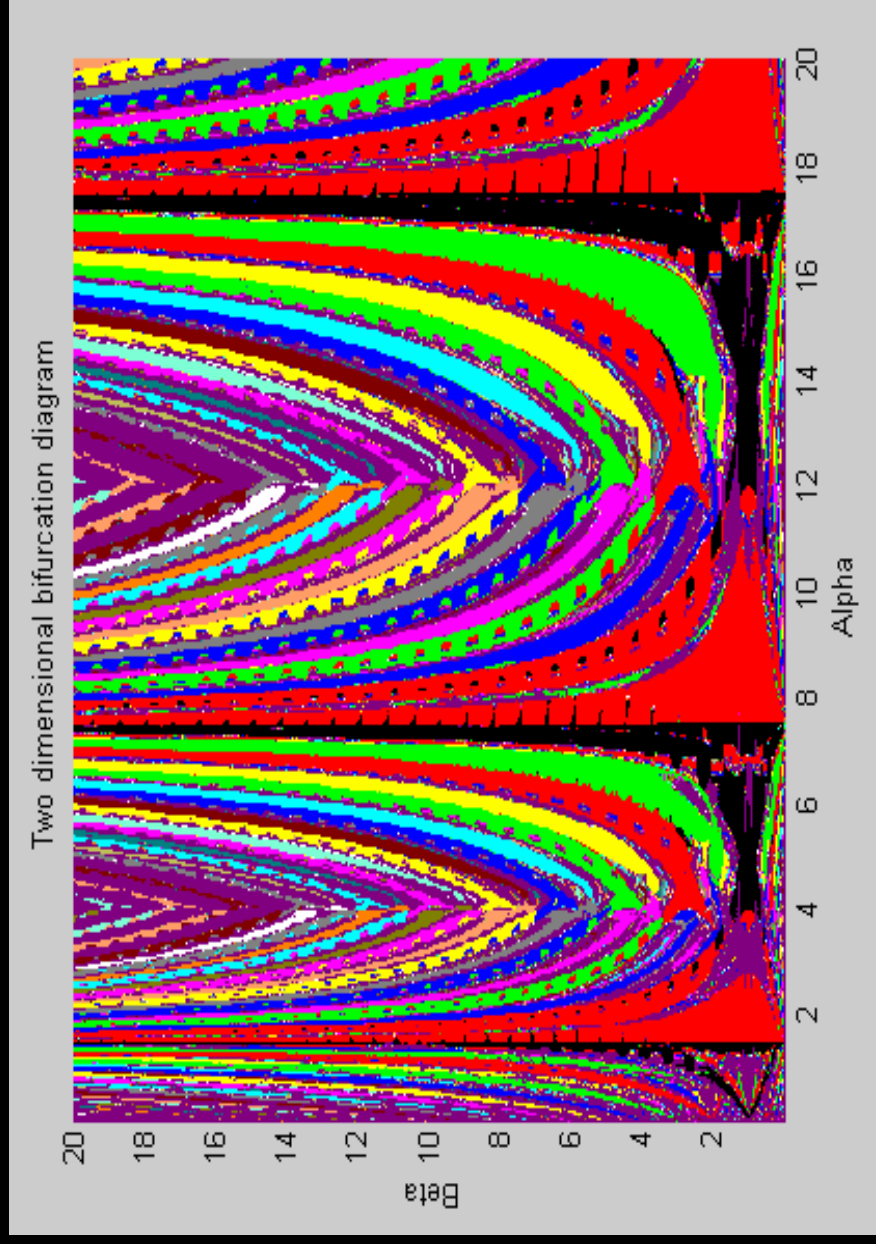
Bifurcations in a two-block stick-slip system



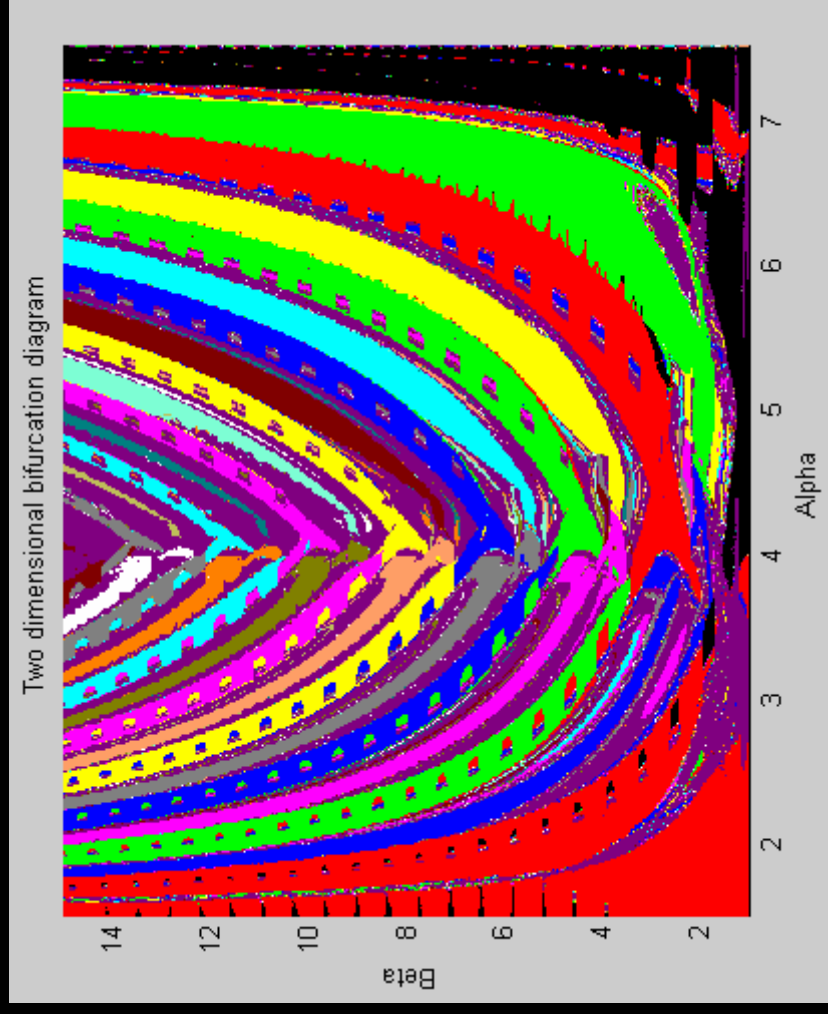
Bifurcations in a two-block stick-slip system



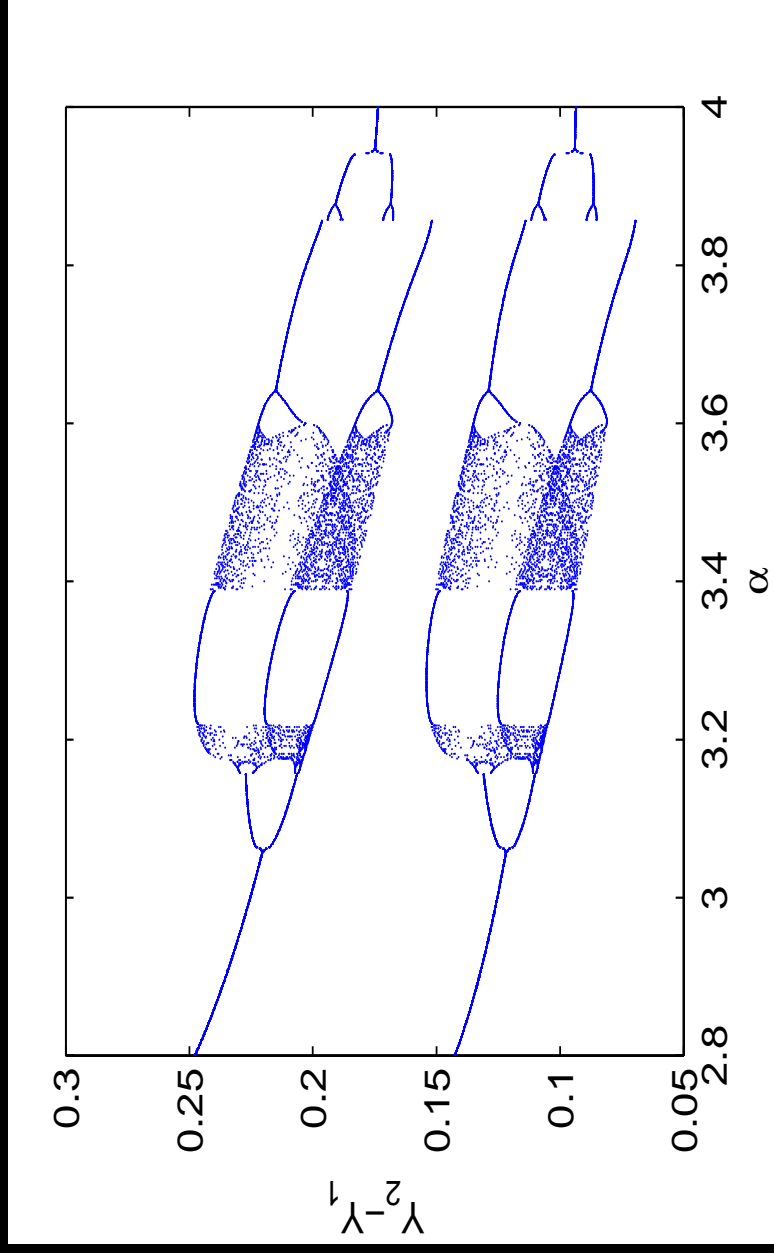
Bifurcations in a two-block stick-slip system



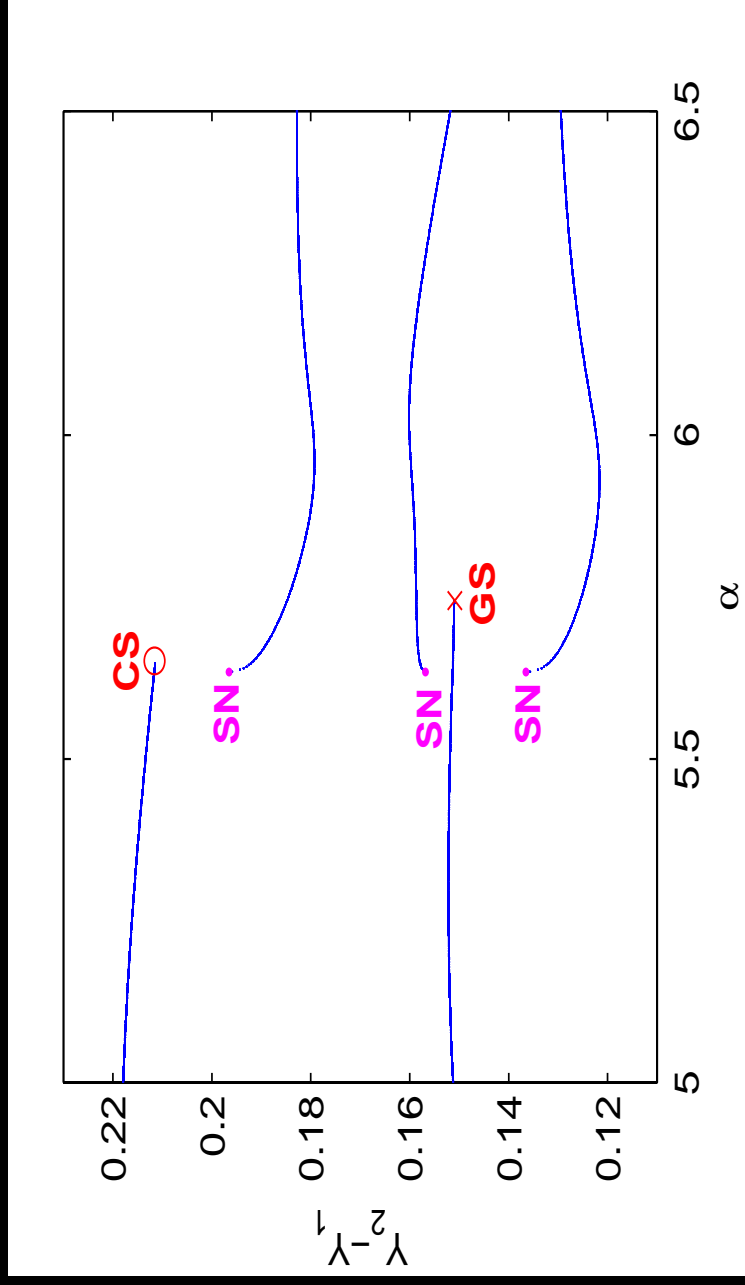
Bifurcations in a two-block stick-slip system



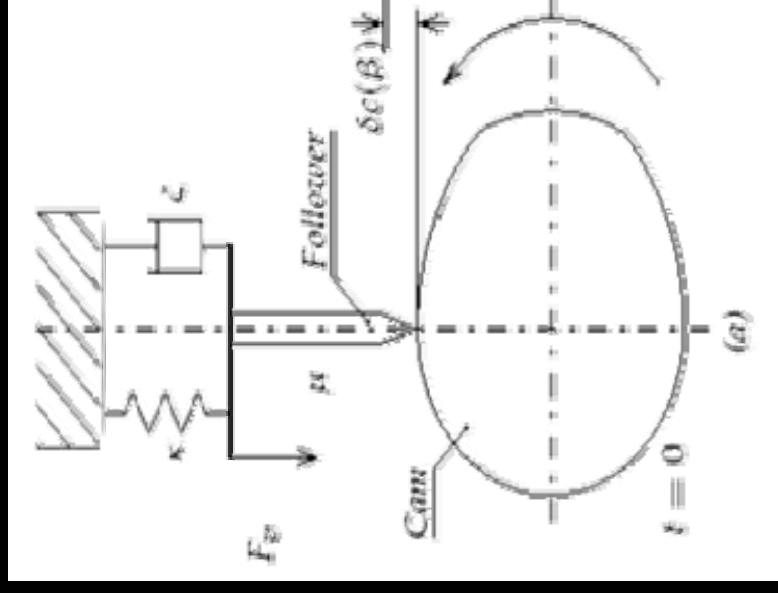
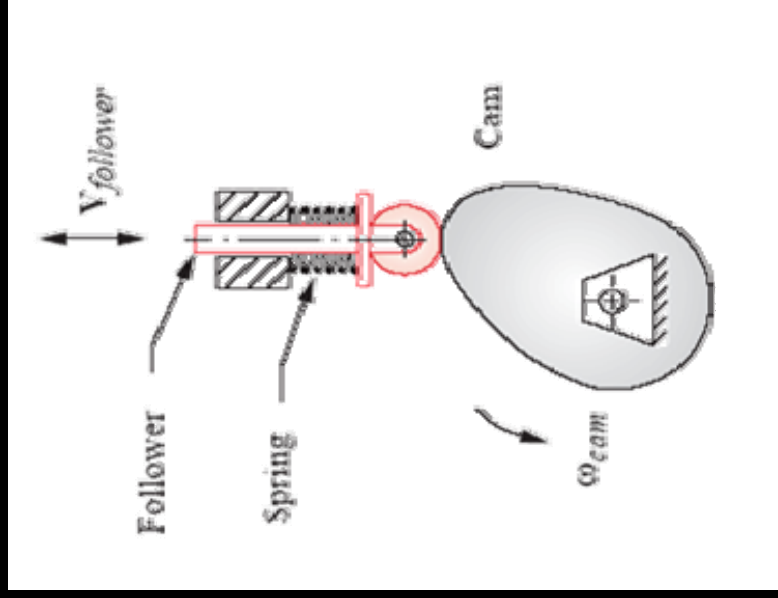
Bifurcations in a two-block stick-slip system



Bifurcations in a two-block stick-slip system



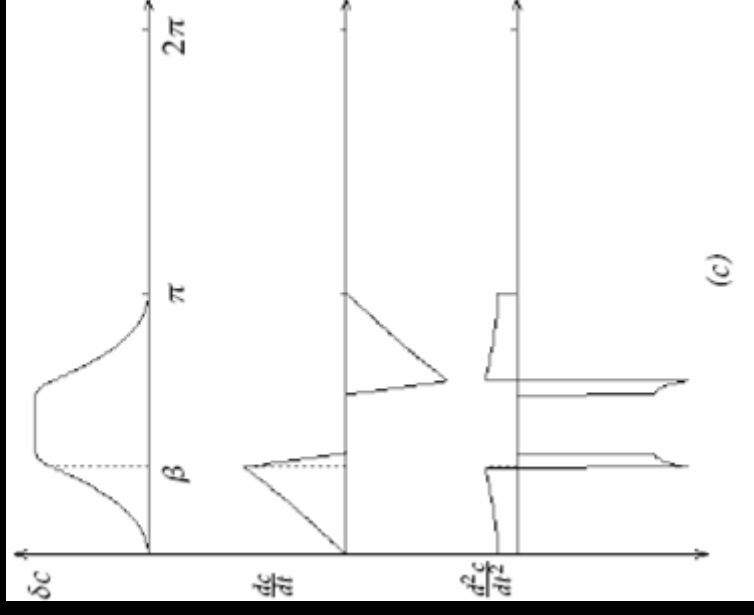
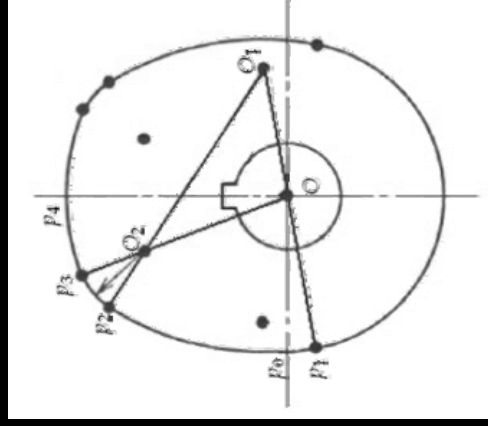
Bifurcations in a Cam Follower System



Cam Profile

The cam is PWS model because:

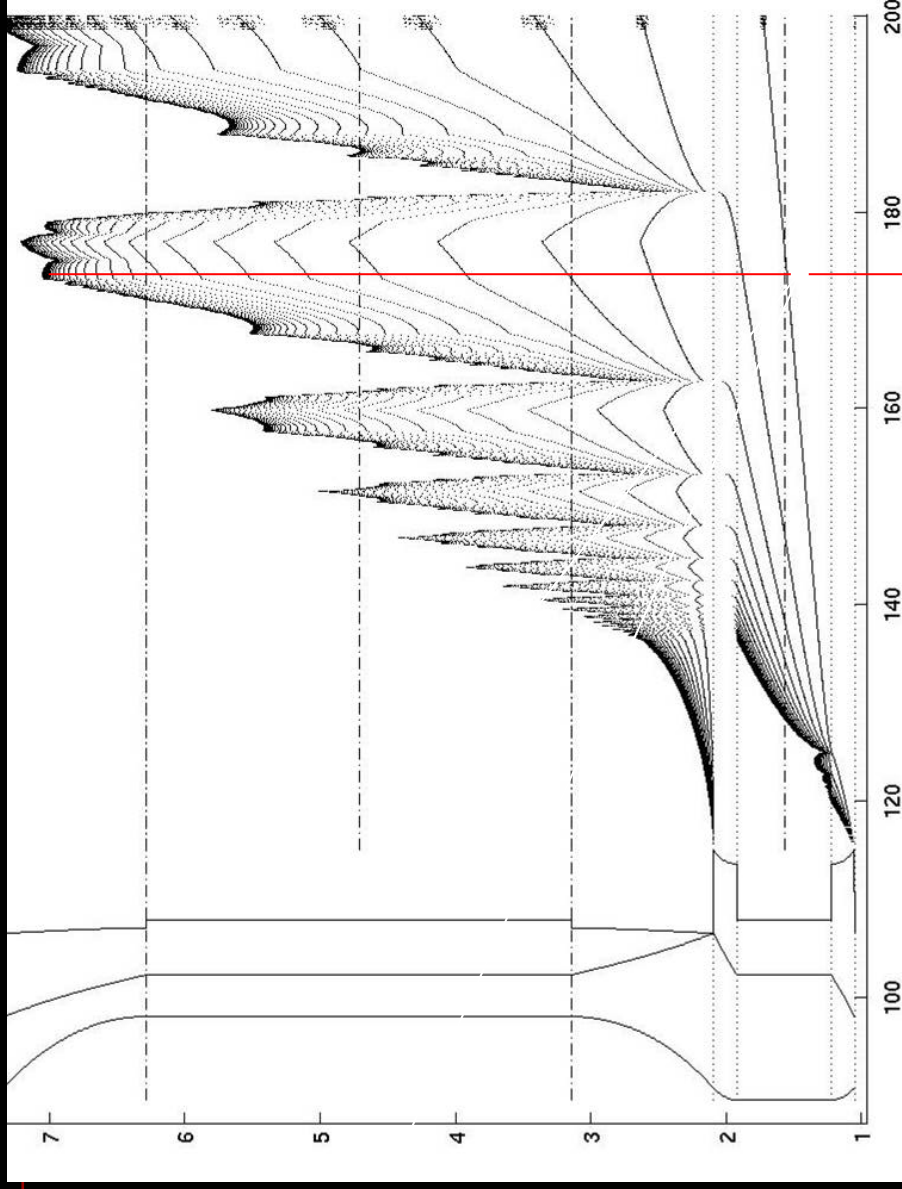
- There are impacts
- The discontinuous nature on the cam profile second derivative



$$c(\beta) = \begin{cases} R_b \sqrt{1 - \frac{k_1^2 c^2 \beta \beta_1}{\rho_1^2}} & \text{If } 0 < \beta \leq \beta_1 \\ -k_1 s \beta \beta_1 + \rho_1 \sqrt{1 - \frac{k_1^2 c^2 \beta \beta_1}{\rho_1^2}} & \text{If } \beta_1 < \beta \leq \beta_2 \\ k_2 s \beta \beta_3 + \rho_3 \sqrt{1 - \frac{k_2^2 c^2 \beta \beta_3}{\rho_3^2}} & \text{If } \beta_2 < \beta \leq \beta_3 \\ \rho_2 & \text{If } \beta_3 < \beta \leq \pi \end{cases}$$

Bifurcation Analysis - Chattering orbits

Phase of
Impact

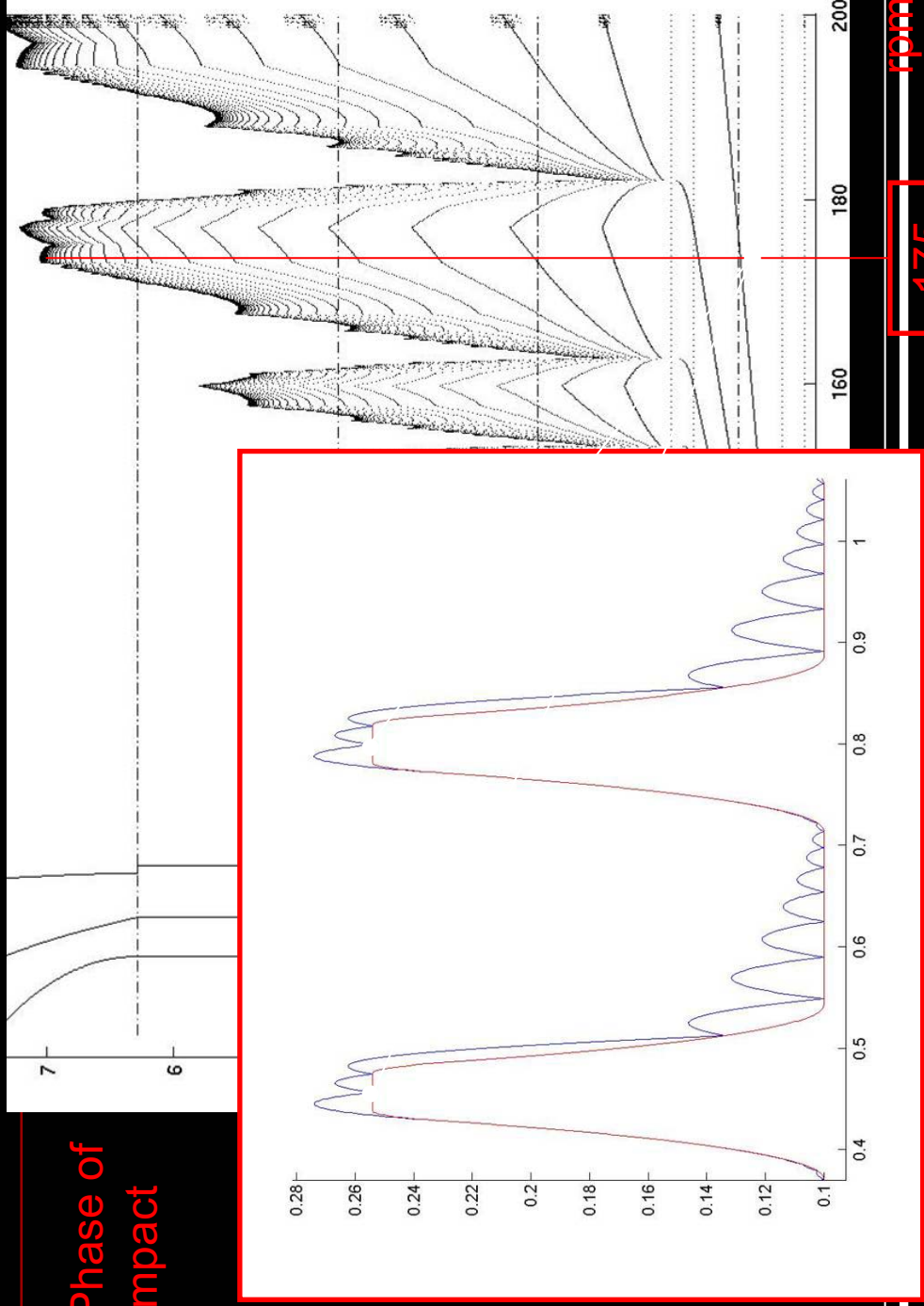


rpm

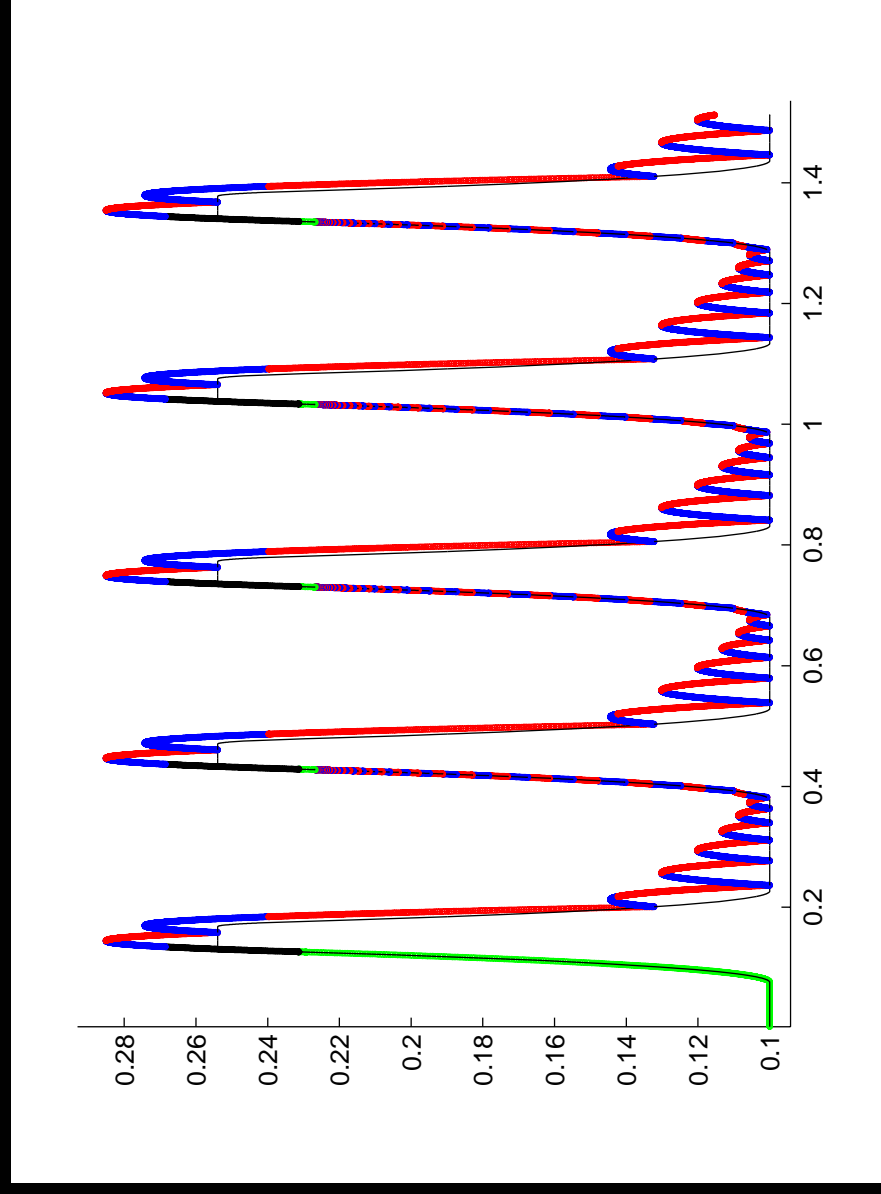
175

Bifurcation Analysis - Chattering orbits

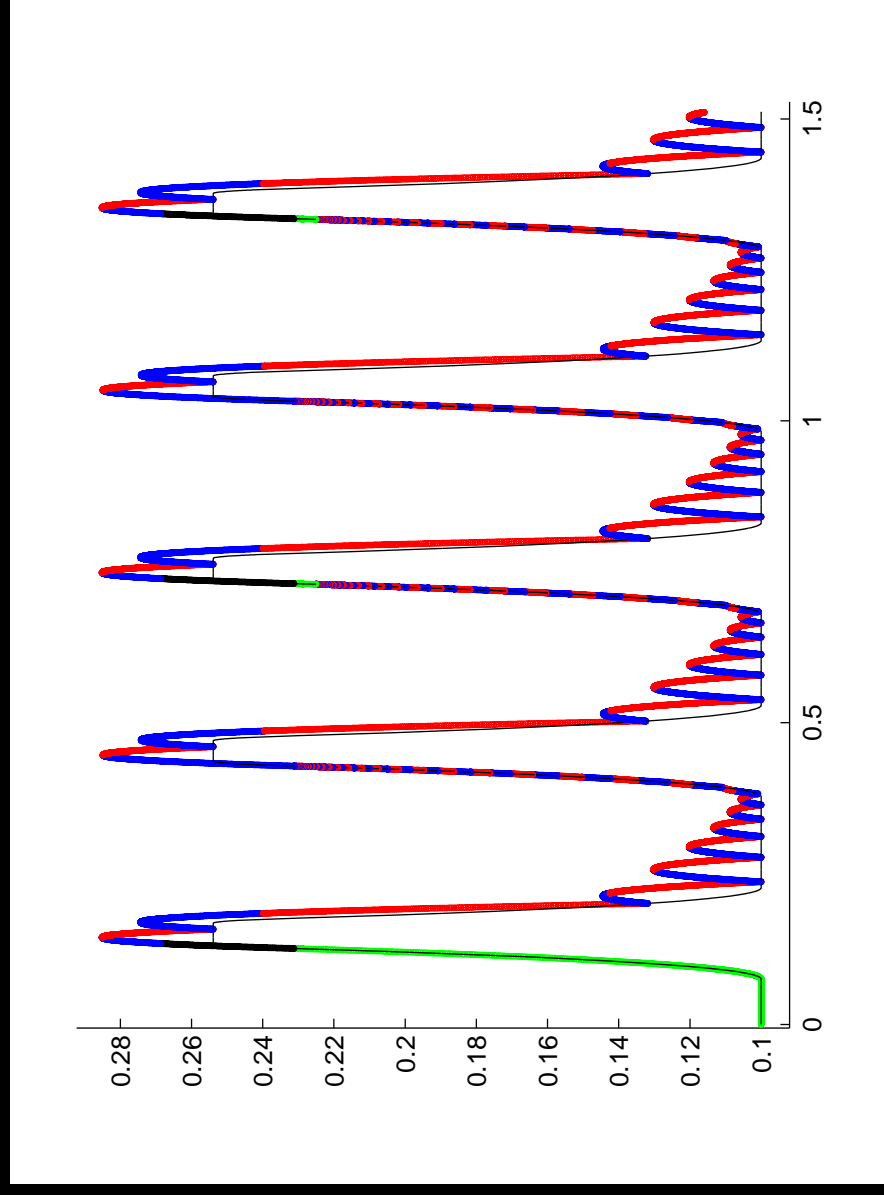
Phase of Impact



Transition from complete chattering to no-complete chattering

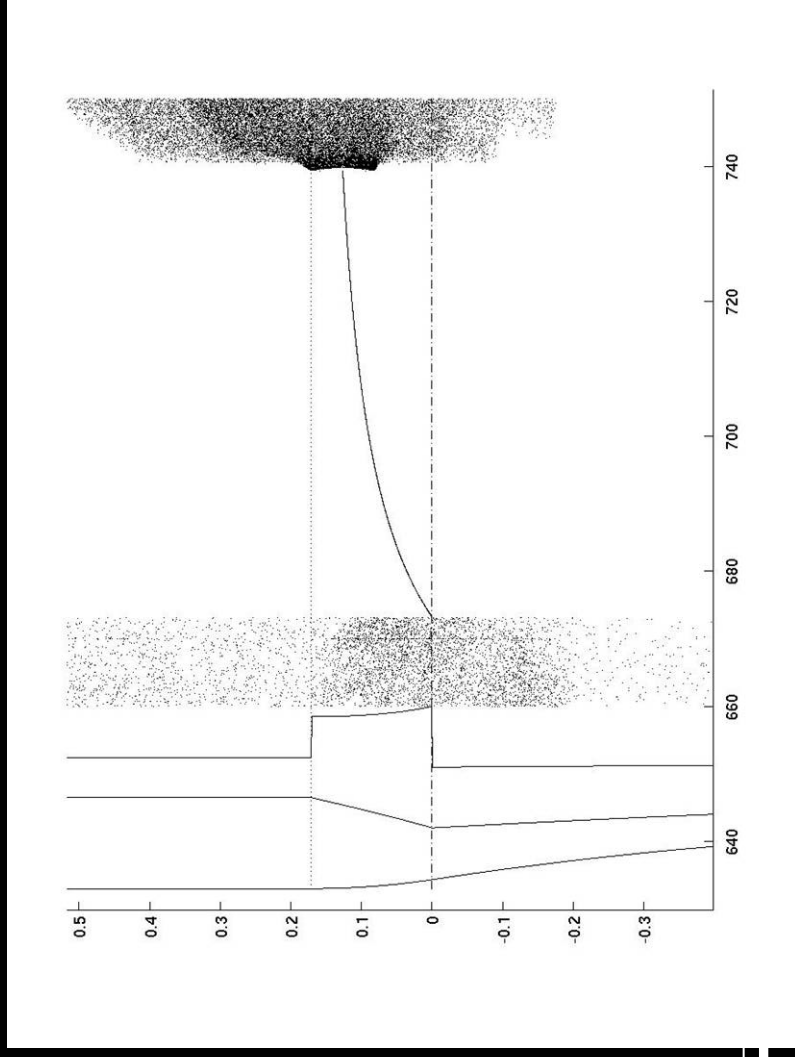


Transition from complete chattering to no-complete chattering



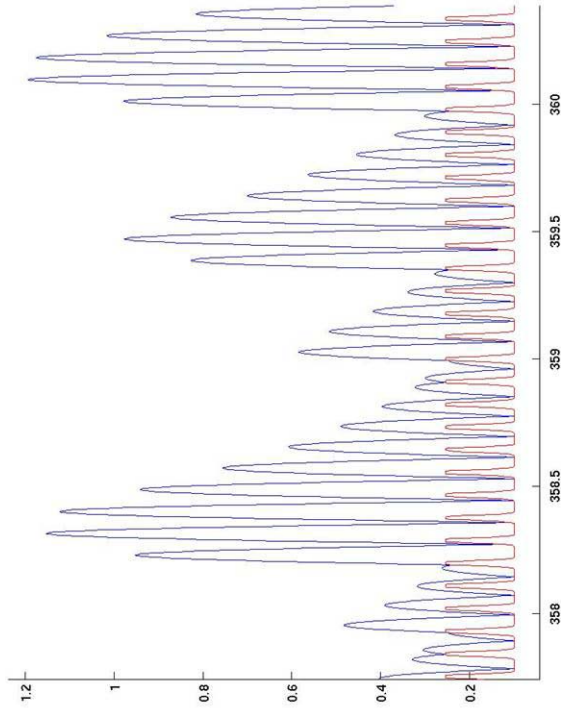
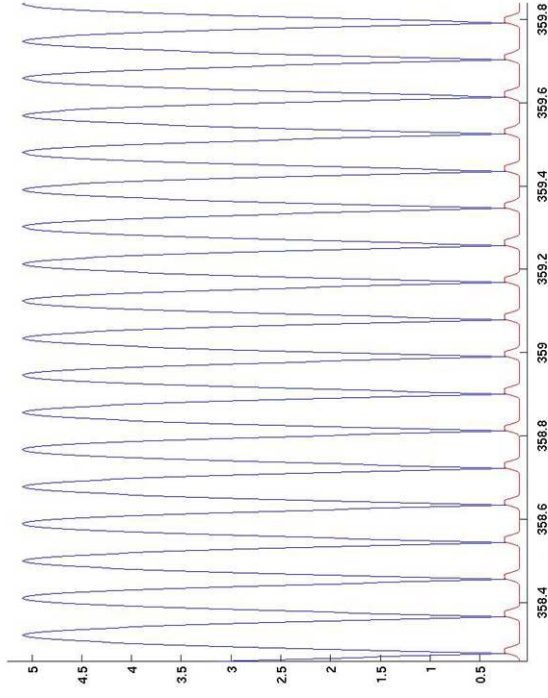
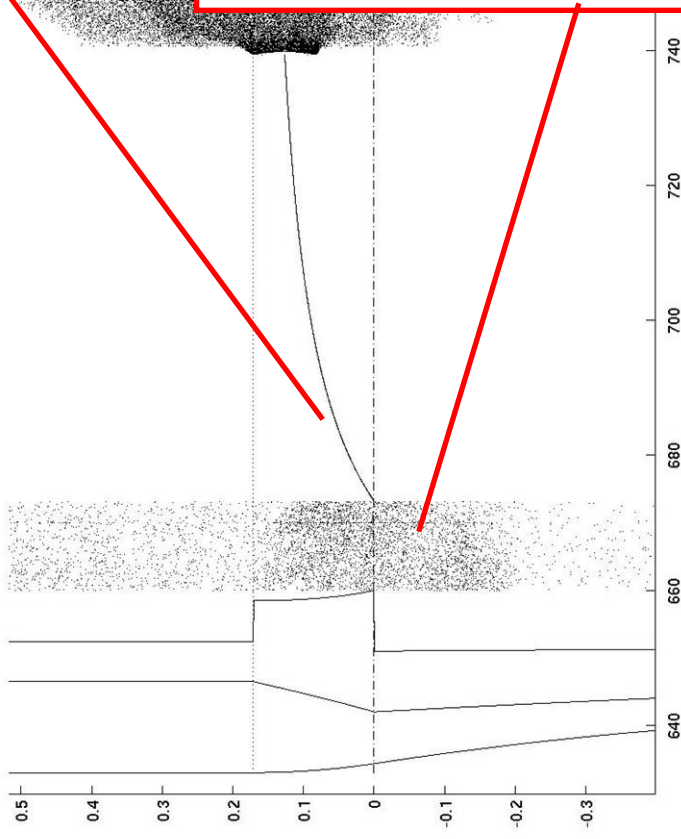
NonSmooth Bifurcation Scenario

Corner Impact Bifurcation

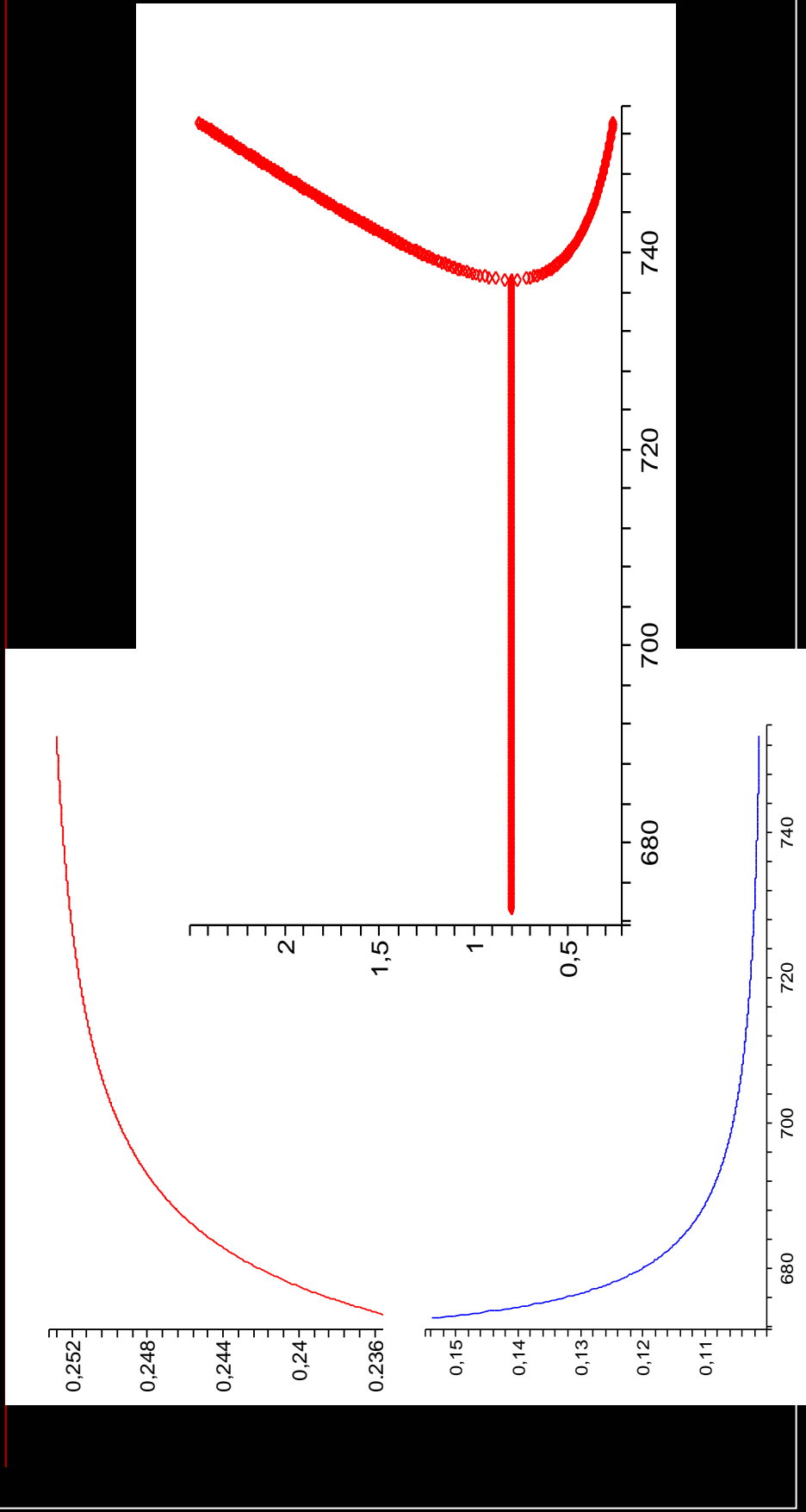


NonSmooth Bifurc

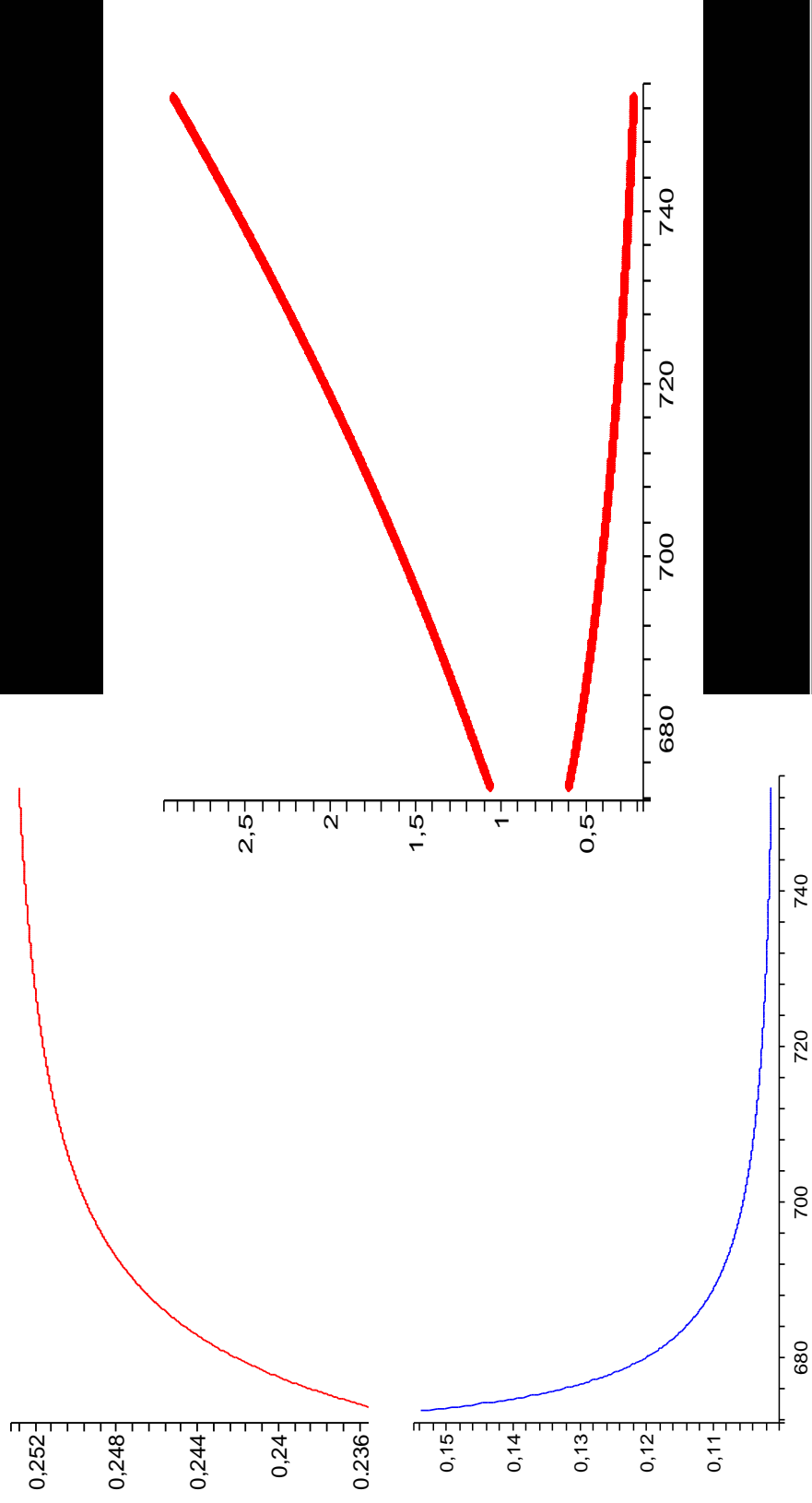
Corner Impact Bifurcation



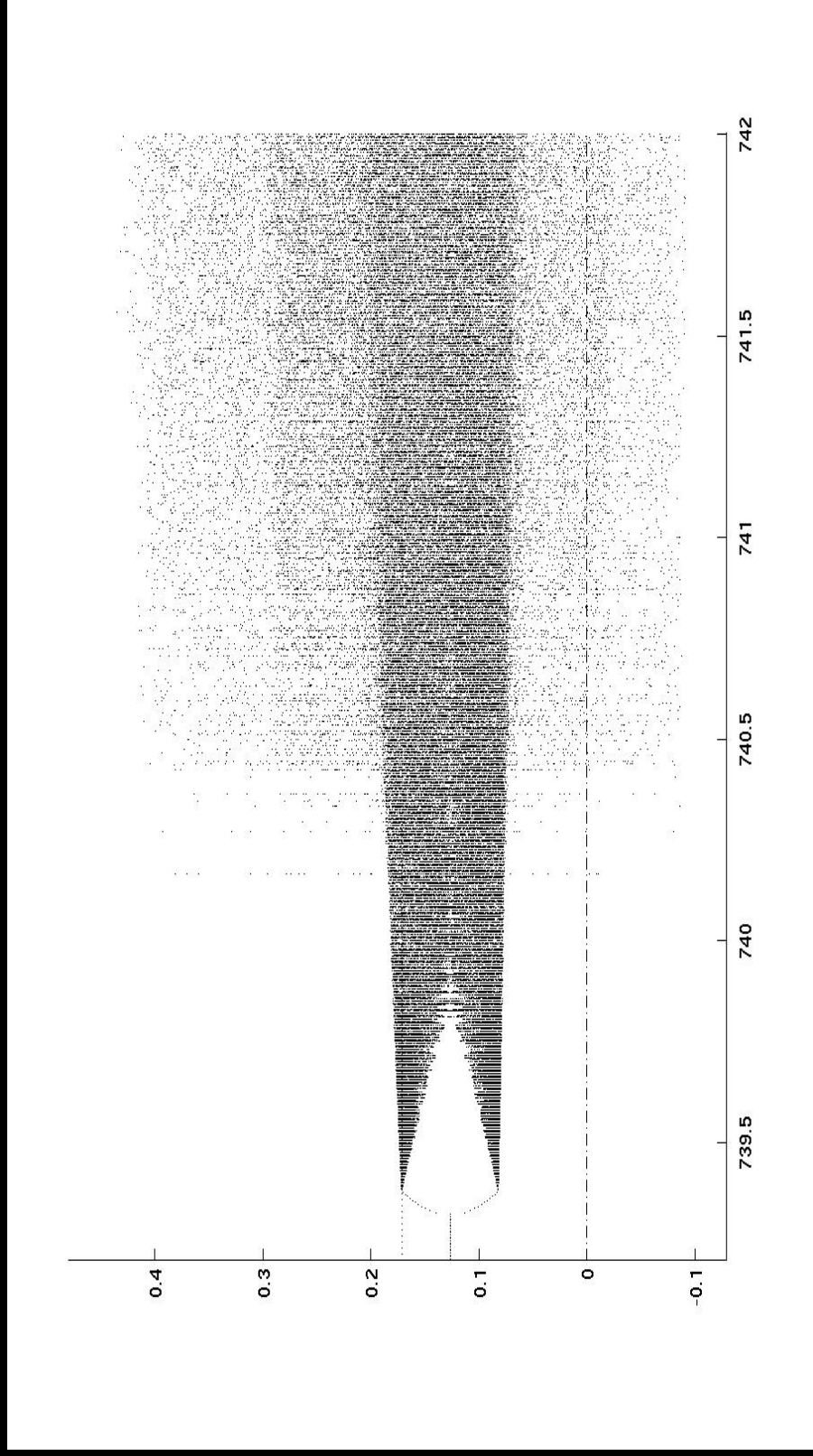
Continuation of periodic orbits



Continuation of periodic orbits



Corner-impact bifurcation in a P(2,1:1)



GRACIAS POR SU ATENCIÓN